

BPhO

Computational Challenge

Thermodynamics

Dr Andrew French.
December 2023.



Engines

The Inner Workings of Machines That Move the World



Theodore Gray

Bestselling author of *The Elements* and *How Things Work*
 Photographs by Nick Mann



THEODORE GRAY
ABC Elements
 The Elements and the Architecture of Everything

THEODORE GRAY
Molecules
 The Elements and the Architecture of Everything

THEODORE GRAY
Reactions
 An Illustrated Exploration of Elements, Molecules, and Changes in the Universe

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TOOLS
 A Visual Exploration of Instruments and Devices in the Industry

THEODORE GRAY
HOW THINGS WORK
 INNER LIFE OF EVERYDAY MACHINES

These books are amazing!

$$Q_{in} = Q_{out} + W \quad \text{1st law}$$

$$\Delta S_H = -\frac{Q_{in}}{T_H} \quad \Delta S_C = \frac{Q_{out}}{T_C} \quad \text{Entropy}$$

$$\Delta S_{total} = \Delta S_H + \Delta S_C = -\frac{Q_{in}}{T_H} + \frac{Q_{out}}{T_C}$$

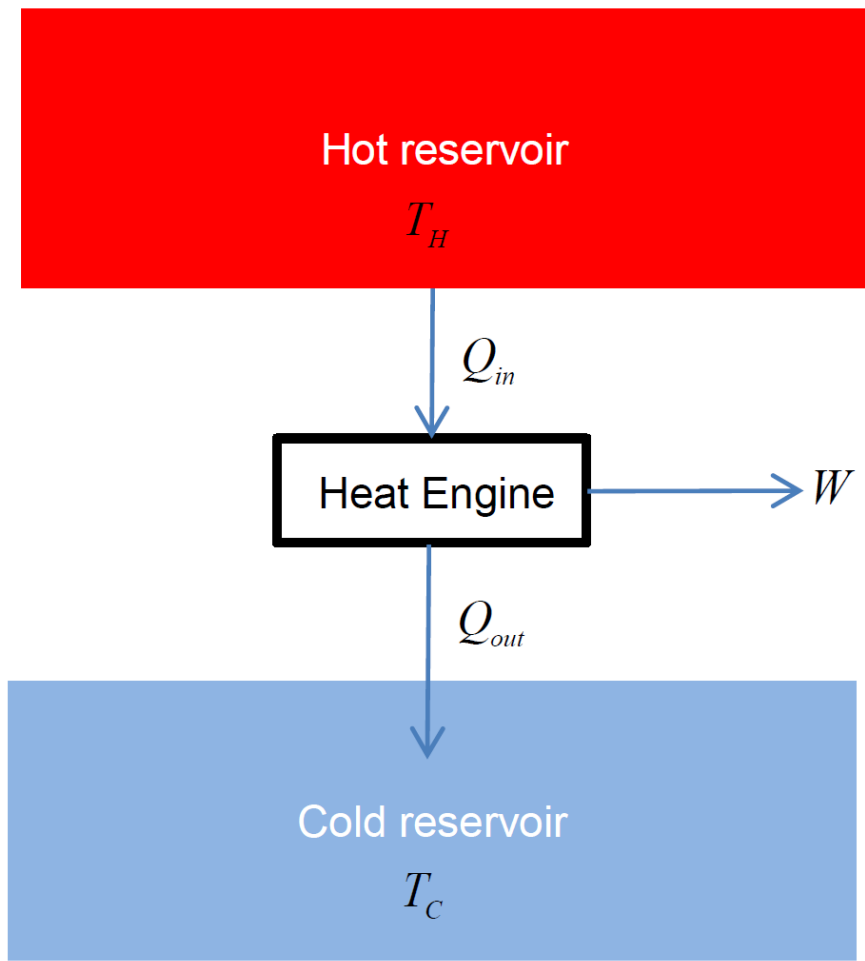
$$\Delta S_{total} \geq 0 \quad \therefore -\frac{Q_{in}}{T_H} + \frac{Q_{out}}{T_C} \geq 0$$

2nd law

$$-\frac{Q_{in}}{T_H} + \frac{Q_{in} - W}{T_C} \geq 0$$

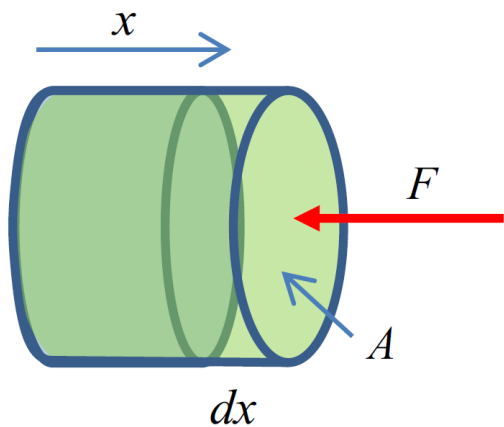
$$\therefore -\frac{1}{T_H} + \frac{1 - \frac{W}{Q_{in}}}{T_C} \geq 0$$

$$-\frac{T_C}{T_H} + 1 - \frac{W}{Q_{in}} \geq 0$$



$$\eta = \frac{W}{Q_{in}} \quad \text{Engine efficiency}$$

$$\therefore -\frac{T_C}{T_H} + 1 - \eta \geq 0 \quad \therefore \eta \leq 1 - \frac{T_C}{T_H}$$



volume change

$$dV = -A dx$$

$$p = \frac{F}{A}$$

The pressure acting upon the gas

$$dW = F dx \quad \text{Work done on the gas}$$

$$\therefore dW = pA \times -\frac{dV}{A}$$

$$\therefore dW = -pdV$$

If heat dQ is supplied to the gas then the **First Law of Thermodynamics** (that Energy in a closed system is conserved) means the internal energy change is

$$dU = dQ - pdV$$

Internal energy

Heat

Work

The internal energy for n moles of an ideal gas is:

$$U = \frac{1}{2} \alpha n R T$$

Molar gas constant
8.314 Jmol⁻¹K⁻¹

Temperature
(Kelvin K)

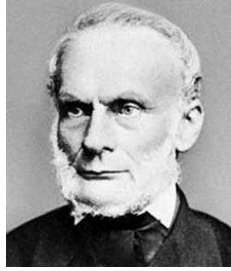
Internal energy of n moles of gas

Number of degrees of freedom of molecular motion (e.g $\alpha = 3$ for x, y, z translation)

IDEAL GAS EQUATION

$$pV = nRT$$

$$dS = \frac{dQ}{T}$$



Rudolf Clausius
1822-1888

ISOTHERMAL PROCESS

$$dT = 0$$

$$pV = \text{constant}$$

$$Q = W = nRT \ln \left(\frac{V}{V_0} \right)$$

Heat input = work done by gas since no change in U

ISENTROPIC (CONSTANT ENTROPY) PROCESS

$$dS = dQ = 0 \quad \text{No heat exchanged}$$

$$pV^\gamma = \text{constant}$$

$$\gamma = \frac{c_P}{c_V} = 1 + \frac{2}{\alpha}$$

Ratio of constant pressure to constant volume process heat capacities

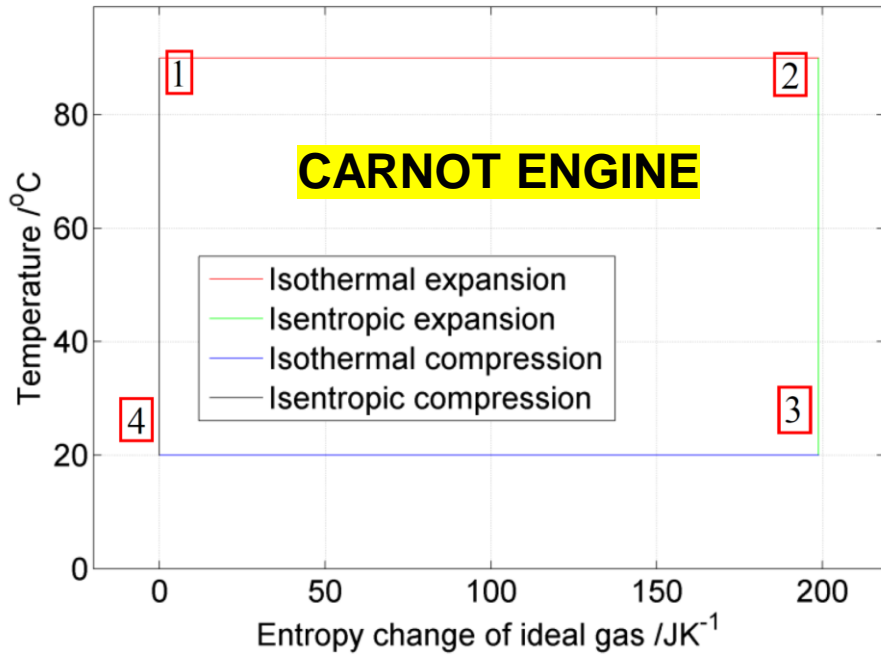
$$W = \frac{p_0 V_0}{\gamma - 1} \left(1 - \left(\frac{V_0}{V} \right)^{\gamma - 1} \right)$$

Work done by gas on the surroundings

$$c_V = \frac{R}{M} \frac{1}{\gamma - 1}$$

$$c_P = \frac{R}{M} \frac{\gamma}{\gamma - 1}$$

Carnot cycle: $T_H = 90^\circ\text{C}$, $T_C = 20^\circ\text{C}$, $V_1 = 0.1\text{m}^3$
 $W = 13.9266\text{kJ}$, $Q_{\text{in}} = 72.2194\text{kJ}$, Efficiency = $W/Q_{\text{in}} = 19.2837\%$



From the **First Law of Thermodynamics**

$$dU = dQ - pdV \quad \therefore dU = Tds - pdV$$

Over the whole cycle the *internal energy doesn't change*, so the work done by the gas is

$$W = \oint pdV = \oint Tds$$

i.e. area enclosed by cycle

$$W = (T_H - T_C) \Delta S = (T_H - T_C) nR \ln \left(\frac{V_2}{V_1} \right)$$

$$\Delta S = \frac{Q_{\text{in}}}{T_H} = \frac{Q_{\text{out}}}{T_C} = nR \ln \left(\frac{V_2}{V_1} \right)$$

ISOTHERMAL PROCESS

$$dT = 0$$

$$pV = \text{constant}$$

$$Q = W = nRT \ln \left(\frac{V}{V_0} \right)$$

Heat input = work done by gas
 Since no change in U

$$\therefore \eta = \frac{W}{Q_{\text{in}}} = \frac{(T_H - T_C) nR \ln \left(\frac{V_2}{V_1} \right)}{nRT_H \ln \left(\frac{V_2}{V_1} \right)} = 1 - \frac{T_C}{T_H}$$

$$\eta \leq 1 - \frac{T_C}{T_H}$$

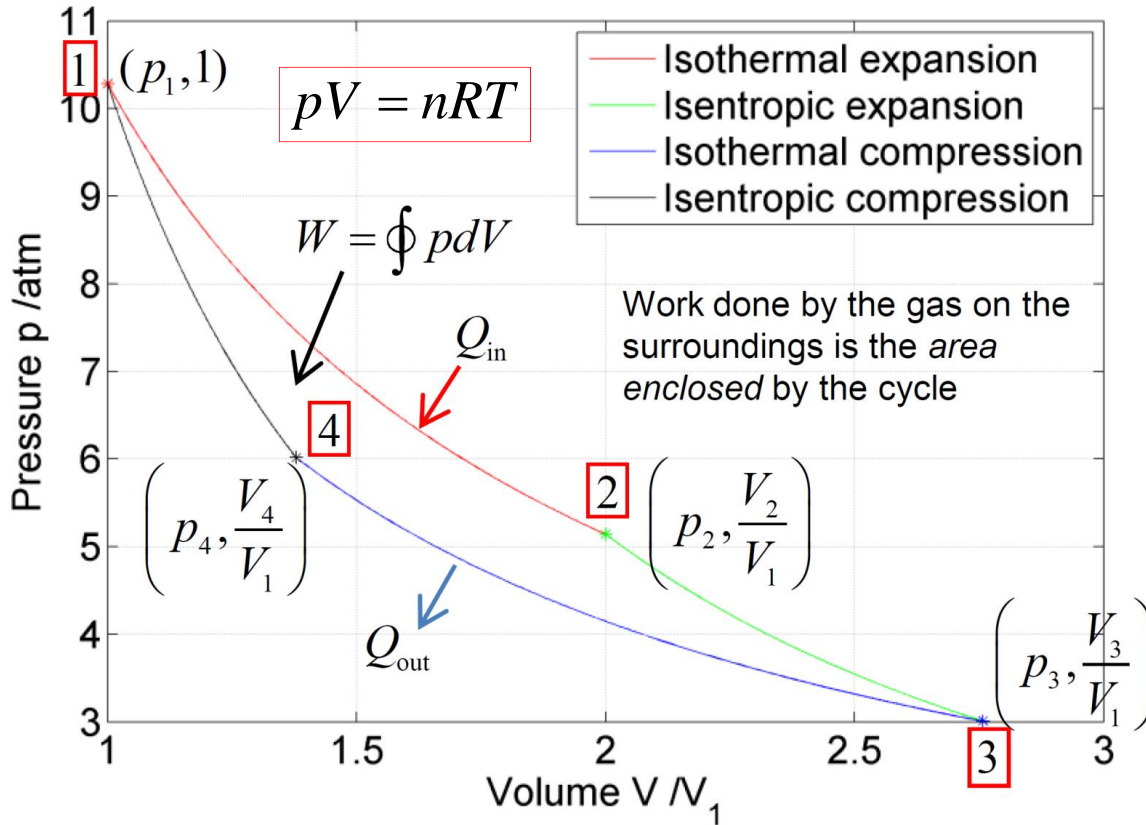
So Carnot engine is (one example) of the *most efficient heat engine possible*.

Carnot cycle: $T_H = 90^\circ\text{C}$, $T_C = 20^\circ\text{C}$, $V_1 = 0.1\text{m}^3$

$W = 13.9266\text{kJ}$, $Q_{in} = 72.2194\text{kJ}$, Efficiency = $W/Q_{in} = 19.2837\%$



Nicolas Léonard Sadi Carnot (1796-1832)



$$W = (T_H - T_C) nR \ln \left(\frac{V_2}{V_1} \right)$$

$$Q_{in} = nRT_H \ln \left(\frac{V_2}{V_1} \right)$$

$$Q_{out} = nRT_C \ln \left(\frac{V_3}{V_4} \right)$$

Inputs

$$T_H, T_C, V_1, V_2, n$$

$$V_4 = \left(\frac{T_H}{T_C} \right)^{\frac{1}{\gamma-1}} V_1$$

$$V_3 = \left(\frac{T_H}{T_C} \right)^{\frac{1}{\gamma-1}} V_2$$

Next step: code up a Heat Cycle model! Start with a spreadsheet, then try MATLAB/Python etc ...The key idea is to VISUALIZE your solutions.

Carnot Cycle model

Dr A. French. September 2017



Nicolas
Léonard Sadi
Carnot
(1796-1832)

Input parameters

Hot reservoir temperature /Celsius	150
Cold reservoir temperature /Celsius	20
Mass of gas /g	1.00
Volume of gas at lowest volume and highest pressure /litres	0.40
Volume of gas after isothermal expansion /litres	1.00
Degrees of freedom of molecular motion	3
Molar mass of gas /gmol ⁻¹	28.966

Outputs

Heat input during isothermal expansion /kJ	0.111
Heat output during isothermal compression /kJ	0.077
Total work done by gas on surroundings /kJ	0.034
Entropy change during isothermal stages /JK ⁻¹	0.263
Efficiency (work done / heat input)	0.307

Theoretical efficiency

	0.307
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$$\eta = 1 - \frac{T_c}{T_h}$$

Note all temperatures incorporated into calculations will be converted to Kelvin first - i.e. add 273 to Celsius number.

Pressure, volume coordinates of heat cycle

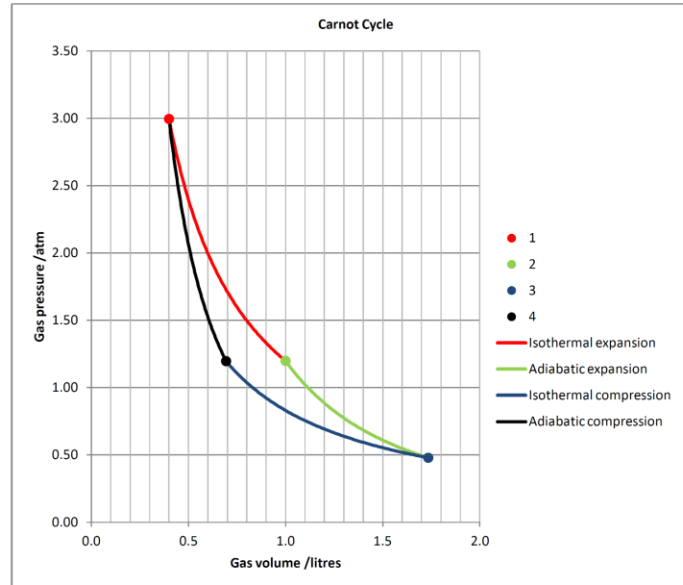
p1	3.00
V1	0.4
p2	1.20
V2	1.0
p3	0.48
V3	1.73
p4	1.20
V4	0.7

Note all pressures are quoted in atmospheres. 1atm = 101,325 Pa. Volumes in litres. T in K.

Reservoir temperatures in K

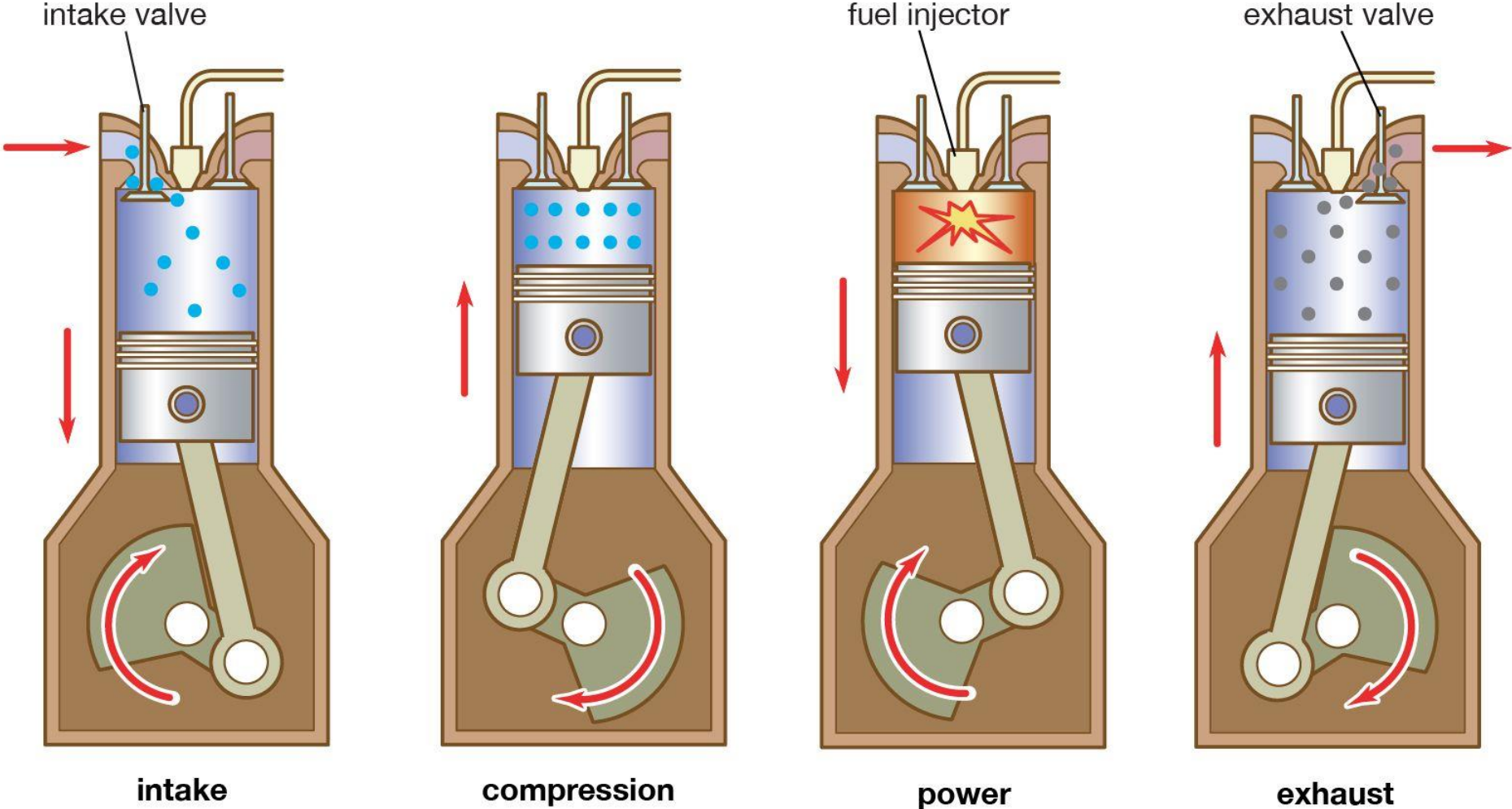
T_H	423
T_C	293

Number of moles of gas in engine	0.035
Ratio of specific heats gamma	1.667

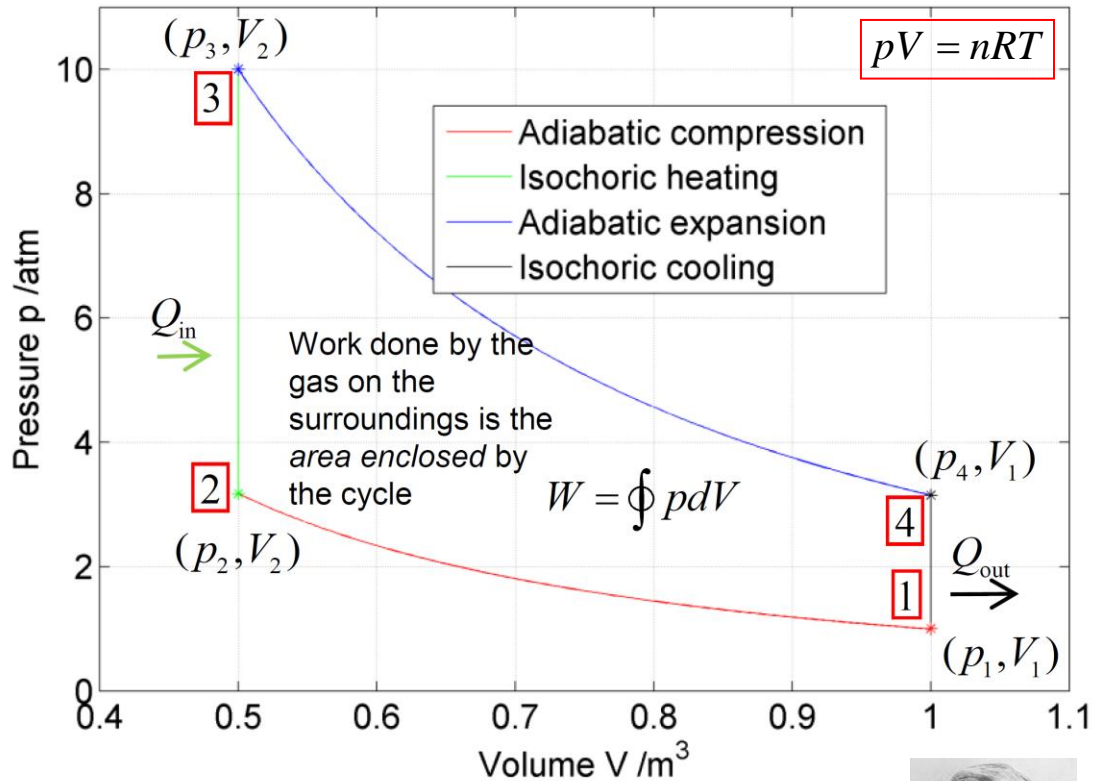


V diff fraction	1 to 2		2 to 3		3 to 4		4 to 1	
	Isothermal expansion p	Isothermal expansion V	Adiabatic expansion p	Adiabatic expansion V	Isothermal compression p	Isothermal compression V	Adiabatic compression p	Adiabatic compression V
0	2.996	0.400	1.198	1.000	0.478	1.735	1.196	0.694
0.01	2.951	0.406	1.184	1.007	0.481	1.724	1.205	0.691
0.02	2.908	0.412	1.169	1.015	0.484	1.714	1.213	0.688
0.03	2.867	0.418	1.155	1.022	0.487	1.703	1.222	0.685
0.04	2.826	0.424	1.142	1.029	0.490	1.693	1.231	0.682
0.05	2.787	0.430	1.128	1.037	0.493	1.683	1.240	0.679
0.06	2.748	0.436	1.115	1.044	0.496	1.672	1.249	0.676
0.07	2.711	0.442	1.102	1.051	0.499	1.662	1.258	0.673
0.08	2.675	0.448	1.089	1.059	0.503	1.651	1.267	0.670
0.09	2.639	0.454	1.077	1.066	0.506	1.641	1.276	0.667
0.1	2.605	0.460	1.065	1.073	0.509	1.631	1.286	0.664
0.11	2.571	0.466	1.053	1.081	0.512	1.620	1.295	0.662
0.12	2.539	0.472	1.041	1.088	0.516	1.610	1.305	0.659
0.13	2.507	0.478	1.029	1.096	0.519	1.599	1.315	0.656
0.14	2.476	0.484	1.018	1.103	0.522	1.589	1.324	0.653
0.15	2.445	0.490	1.007	1.110	0.526	1.579	1.334	0.650
0.16	2.416	0.496	0.996	1.118	0.529	1.568	1.345	0.647
0.17	2.387	0.502	0.985	1.125	0.533	1.558	1.355	0.644
0.18	2.359	0.508	0.974	1.132	0.536	1.547	1.365	0.641
0.19	2.331	0.514	0.964	1.140	0.540	1.537	1.376	0.638
0.2	2.304	0.520	0.953	1.147	0.544	1.526	1.386	0.635
0.21	2.278	0.526	0.943	1.154	0.547	1.516	1.397	0.632
0.22	2.252	0.532	0.933	1.162	0.551	1.506	1.408	0.629
0.23	2.227	0.538	0.924	1.169	0.555	1.495	1.419	0.626
0.24	2.203	0.544	0.914	1.176	0.559	1.485	1.430	0.623
0.25	2.179	0.550	0.905	1.184	0.563	1.474	1.441	0.620
0.26	2.155	0.556	0.895	1.191	0.567	1.464	1.453	0.617
0.27	2.132	0.562	0.886	1.198	0.571	1.454	1.465	0.615
0.28	2.110	0.568	0.877	1.206	0.575	1.443	1.476	0.612
0.29	2.088	0.574	0.868	1.213	0.579	1.433	1.488	0.609
0.3	2.066	0.580	0.860	1.220	0.584	1.422	1.500	0.606
0.31	2.045	0.586	0.851	1.228	0.588	1.412	1.512	0.603
0.32	2.024	0.592	0.843	1.235	0.592	1.402	1.525	0.600
0.33	2.004	0.598	0.834	1.242	0.597	1.391	1.537	0.597
0.34	1.984	0.604	0.826	1.250	0.601	1.381	1.550	0.594
0.35	1.964	0.610	0.818	1.257	0.606	1.370	1.563	0.591
0.36	1.945	0.616	0.810	1.264	0.610	1.360	1.576	0.588
0.37	1.926	0.622	0.803	1.272	0.615	1.350	1.589	0.585
0.38	1.908	0.628	0.795	1.279	0.620	1.339	1.603	0.582
0.39	1.890	0.634	0.787	1.287	0.625	1.329	1.616	0.579
0.4	1.872	0.640	0.780	1.294	0.630	1.318	1.630	0.576

Four-stage engine (modelled by Diesel or Otto cycles)



Otto cycle
 $T_1 = 20^\circ\text{C}$, $T_2 = 192.1085^\circ\text{C}$, $T_3 = 1192^\circ\text{C}$, $T_4 = 649.8922^\circ\text{C}$
 $W = 191.9293\text{kJ}$, $Q_{in} = 518.6724\text{kJ}$, Efficiency = $W/Q_{in} = 37.0039\%$



Inputs

$$p_1, p_3, V_1, V_2, T_1, n$$

Molar mass of gas

$$M_{air} \approx 29\text{gmol}^{-1}$$



Nikolaus Otto
(1832-1891)

$$\eta_{otto} = 1 - \frac{1}{r^{\gamma-1}}$$

$$W = \frac{V_2}{\gamma-1} \left(1 - \frac{1}{r^{\gamma-1}} \right) \{ p_3 - p_1 r^\gamma \}$$

$$r = \frac{V_1}{V_2}$$

$$Q_{out} = Q_{in} - W$$

$$Q_{in} = \frac{V_2 (p_3 - p_1 r^\gamma)}{\gamma - 1}$$



FORD MOTOR CO. 1.0L ECOBOOST DOHC DI I-3

Displacement:
999 cc

Block / head material:
cast iron / aluminum

Horsepower (SAE net):
123 @ 6,000 rpm

Torque:
125 lb.-ft. (169 Nm)
@ 2,500 rpm

Specific output:
123 hp/L

Bore x stroke:
71.9 x 82 mm

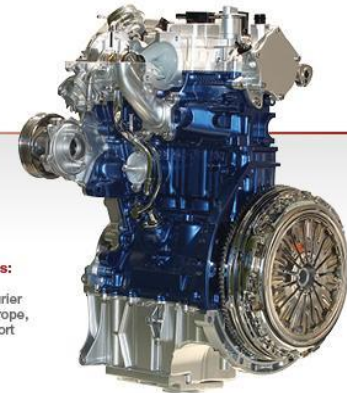
Compression ratio:
10.0:1

EPA city / highway:
31 / 43 mpg

Assembly site:
Cologne, Germany;
Craiova, Romania

Application tested:
'14 Ford Fiesta SE

Additional applications:
Ford Focus, C-Max,
Mondeo, Transit Courier
(Europe); B-Max (Europe,
Asia/Pacific); EcoSport
(Asia/Pacific)



Otto Cycle model
Dr A. French, September 2017



Nikolaus Otto
(1832-1891)

Input parameters	
Temperature T1 of air draw into piston /Celsius	20
Low pressure state p1 /atm	1.00
High pressure state p3 /atm	100
Volume V1 of uncompressed gas /litres	1
Volume V2 of compressed /litres	0.1
Degrees of freedom of molecular motion	3
Molar mass of gas /gmol ⁻¹	28.966

Outputs	
Heat input during isochoric heating /kJ	0.814
Heat output during isochoric cooling /kJ	0.175
Total work done by gas on surroundings /kJ	0.639
Efficiency (work done / heat input)	0.785

Theoretical efficiency	
	0.785

$$\eta = 1 - \frac{1}{r^{\gamma-1}}$$

$$r = \frac{V_1}{V_2} \quad \gamma = \frac{c_p}{c_v}$$

Note all temperatures incorporated into calculations will be converted to Kelvin first - i.e. add 273 to Celsius number.

Pressure, volume, temperature coordinates of heat cycle

p1	1.00
V1	1.000
T1	293
p2	46.42
V2	0.100
T2	1360
p3	100.00
V3	0.100
T3	2930
p4	2.15
V4	1.000
T4	631

Power per cylinder

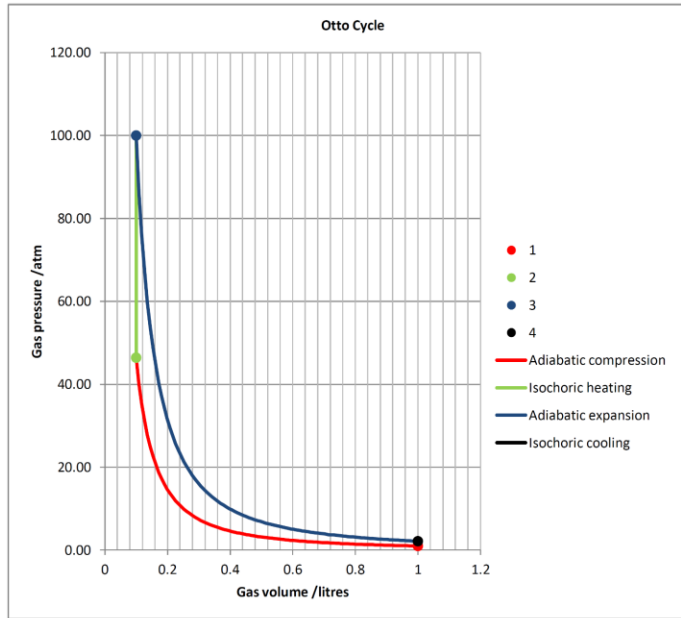
Engine RPM	6500
Power output /kW	69.21979

Number of cylinders

Number of cylinders	1
Total power output /kW	69.22

Note all pressures are quoted in atmospheres. 1atm = 101,325 Pa. Volumes in litres. T in K.

Number of moles of gas in engine	0.042
Ratio of specific heats gamma	1.667
Constant volume specific heat capacity /Jkg ⁻¹ K ⁻¹	431
Constant pressure specific heat capacity /Jkg ⁻¹ K ⁻¹	718



Note real petrol engines have an efficiency of more like 20%, whereas diesels can be up to 40%. In other words, significant losses!

FORD MOTOR CO.
1.0L ECOBOOST
DOHC DI I-3

Displacement:
999 cc

Block / head material:
cast iron / aluminum

Horsepower (SAE net):
123 @ 6,000 rpm

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125 lb.-ft. (169 Nm)
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71.9 x 82 mm

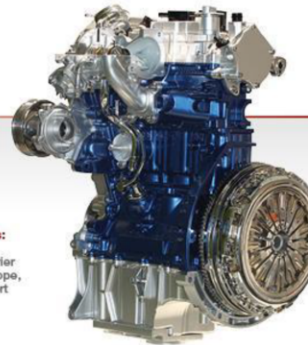
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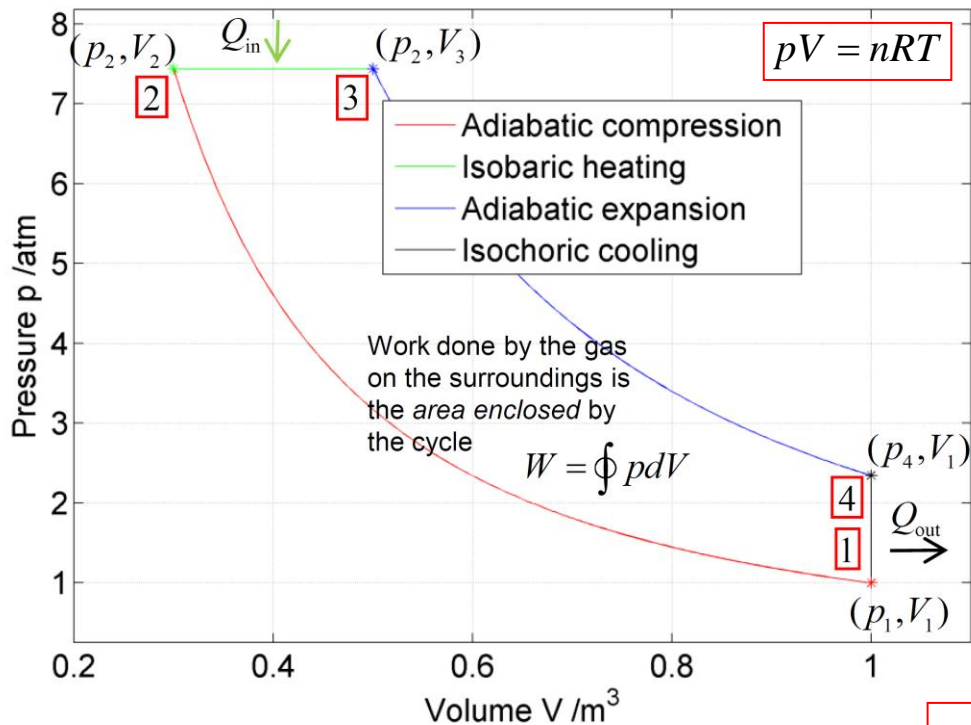
V or p diff fraction	1 to 2		2 to 3		3 to 4		4 to 1	
	Adiabatic compression		Isochoric heating		Adiabatic expansion		Isochoric cooling	
	p	V	p	V	p	V	p	V
0	1.000	1.000	46.416	0.100	100.000	0.100	2.154	1.000
0.01	1.015	0.991	46.952	0.100	86.621	0.109	2.143	1.000
0.02	1.031	0.982	47.488	0.100	75.892	0.118	2.131	1.000
0.03	1.047	0.973	48.023	0.100	67.142	0.127	2.120	1.000
0.04	1.063	0.964	48.559	0.100	59.901	0.136	2.108	1.000
0.05	1.080	0.955	49.095	0.100	53.834	0.145	2.097	1.000
0.06	1.097	0.946	49.631	0.100	48.693	0.154	2.085	1.000
0.07	1.115	0.937	50.167	0.100	44.295	0.163	2.074	1.000
0.08	1.133	0.928	50.703	0.100	40.500	0.172	2.062	1.000
0.09	1.151	0.919	51.238	0.100	37.199	0.181	2.051	1.000
0.1	1.170	0.910	51.774	0.100	34.309	0.190	2.039	1.000
0.11	1.190	0.901	52.310	0.100	31.762	0.199	2.027	1.000
0.12	1.210	0.892	52.846	0.100	29.505	0.208	2.016	1.000
0.13	1.230	0.883	53.382	0.100	27.494	0.217	2.004	1.000
0.14	1.252	0.874	53.918	0.100	25.693	0.226	1.993	1.000
0.15	1.273	0.865	54.454	0.100	24.074	0.235	1.981	1.000
0.16	1.296	0.856	54.989	0.100	22.613	0.244	1.970	1.000
0.17	1.319	0.847	55.525	0.100	21.288	0.253	1.958	1.000
0.18	1.343	0.838	56.061	0.100	20.083	0.262	1.947	1.000
0.19	1.367	0.829	56.597	0.100	18.984	0.271	1.935	1.000
0.2	1.392	0.820	57.133	0.100	17.978	0.280	1.924	1.000
0.21	1.418	0.811	57.669	0.100	17.054	0.289	1.912	1.000
0.22	1.444	0.802	58.204	0.100	16.205	0.298	1.900	1.000
0.23	1.472	0.793	58.740	0.100	15.421	0.307	1.889	1.000
0.24	1.500	0.784	59.276	0.100	14.696	0.316	1.877	1.000
0.25	1.529	0.775	59.812	0.100	14.024	0.325	1.866	1.000
0.26	1.559	0.766	60.348	0.100	13.400	0.334	1.854	1.000
0.27	1.590	0.757	60.884	0.100	12.819	0.343	1.843	1.000
0.28	1.622	0.748	61.419	0.100	12.277	0.352	1.831	1.000
0.29								0.00
0.30								0.00
0.31								0.00
0.32								0.00
0.33								0.00
0.34								0.00
0.35								0.00
0.36								0.00
0.37								0.00
0.38								0.00
0.39								0.00
0.40								0.00
0.41	2.154	0.631	68.385	0.100	7.610	0.469	1.681	1.000
0.42	2.206	0.622	68.921	0.100	7.373	0.478	1.670	1.000
0.43	2.261	0.613	69.457	0.100	7.147	0.487	1.658	1.000
0.44	2.317	0.604	69.993	0.100	6.932	0.496	1.646	1.000
0.45	2.376	0.595	70.529	0.100	6.727	0.505	1.635	1.000
0.46	2.437	0.586	71.065	0.100	6.532	0.514	1.623	1.000



Diesel cycle

$$T_1 = 20^\circ\text{C}, T_2 = 380.8128^\circ\text{C}, T_3 = 816.6881^\circ\text{C}, T_4 = 413.4605^\circ\text{C}$$

$$W = 172.7357\text{kJ}, Q_{in} = 376.835\text{kJ}, \text{Efficiency} = W/Q_{in} = 45.8386\%$$



Inputs

$$p_1, V_1, V_2, V_3, T_1, n$$



$$r = \frac{V_1}{V_2}$$

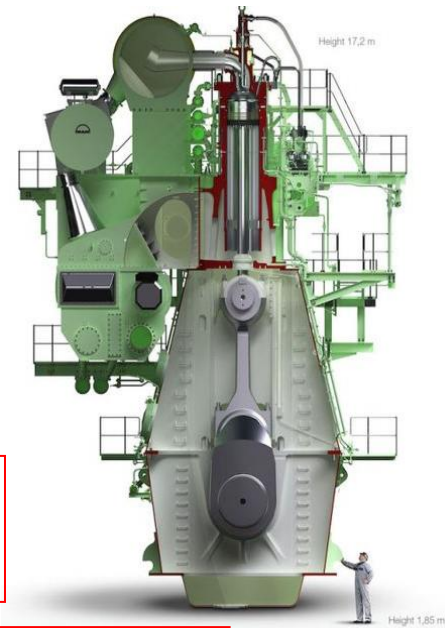
$$s = \frac{V_3}{V_2}$$

$$Q_{out} = Q_{in} - W$$

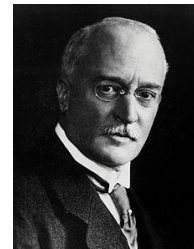
$$Q_{in} = \frac{\gamma}{\gamma - 1} r^\gamma (s - 1) p_1 V_2$$

$$W = \frac{p_1 V_2}{\gamma - 1} \left\{ r(1 - s^\gamma) + r^\gamma (s - 1) \gamma \right\}$$

$$\eta_{diesel} = 1 - \frac{1}{r^{\gamma-1}} \left(\frac{s^\gamma - 1}{\gamma(s - 1)} \right)$$



Rudolf Diesel
(1858-1913)



Diesel Cycle model

Dr A. French, September 2017



Rudolf Diesel
(1858-1913)

Input parameters

Temperature T1 of air draw into piston /Celsius	25
Low pressure state p1 /atm	1.00
Volume V1 of uncompressed gas /litres	1820
Volume V2 of compressed gas /litres	79
Volume V3 of compressed gas after isobaric heating /litres	170
Degrees of freedom of molecular motion	3
Molar mass of gas /g/mol ⁻¹	28.966

Outputs

Heat input during isobaric heating /kJ	4267
Heat output during isochoric cooling /kJ	710
Total work done by gas on surroundings /kJ	3557
Efficiency (work done / heat input)	0.834

Theoretical efficiency

0.834

$$\eta = 1 - \frac{1}{r^{\gamma-1}} \left(\frac{s^{\gamma} - 1}{\gamma(s-1)} \right)$$

$$r = \frac{V_1}{V_2} \quad s = \frac{V_3}{V_2} \quad \gamma = \frac{c_p}{c_v}$$

Note all temperatures incorporated into calculations will be converted to Kelvin first - i.e. add 273 to Celsius number.

Pressure, volume, temperature coordinates of heat cycle

p1	1.0
V1	1820
T1	298
p2	186.0
V2	79.1
T2	2410
p3	186.0
V3	169.7
T3	5168
p4	3.6
V4	1820.0
T4	1063

Single cylinder power output

Engine RPM	84
Power output /kW	4,980

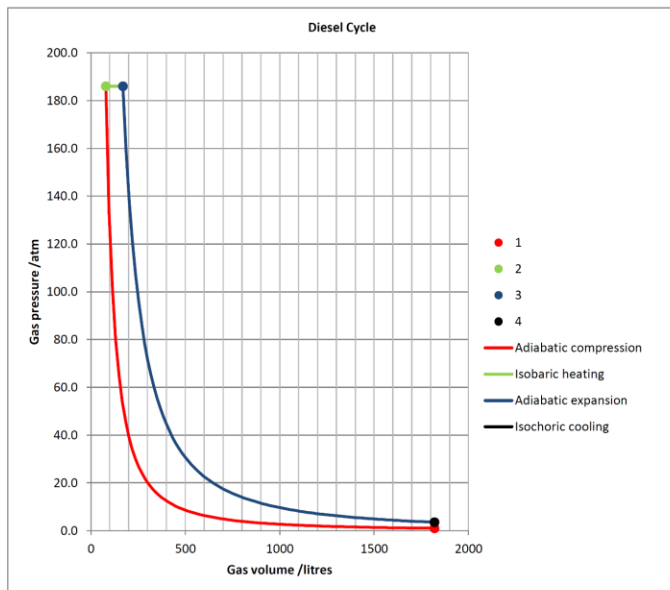
Number of cylinders

Total power output /kW	69,717
------------------------	--------

Note all pressures are quoted in atmospheres. 1atm = 101,325 Pa. Volumes in litres, T in K

Number of moles of gas in engine	74
Ratio of specific heats gamma	1.667
Constant volume specific heat capacity /Jkg ⁻¹ K ⁻¹	431
Constant pressure specific heat capacity /Jkg ⁻¹ K ⁻¹	718

Note real petrol engines have an efficiency of more like 20%, whereas diesels can be up to 40%. In other words, significant losses!



World's largest container ship in 2014
MV CSCL Globe



MAN B&W 12S90ME-C Mark 9.2 diesel engine. 69,720kW at 84RPM

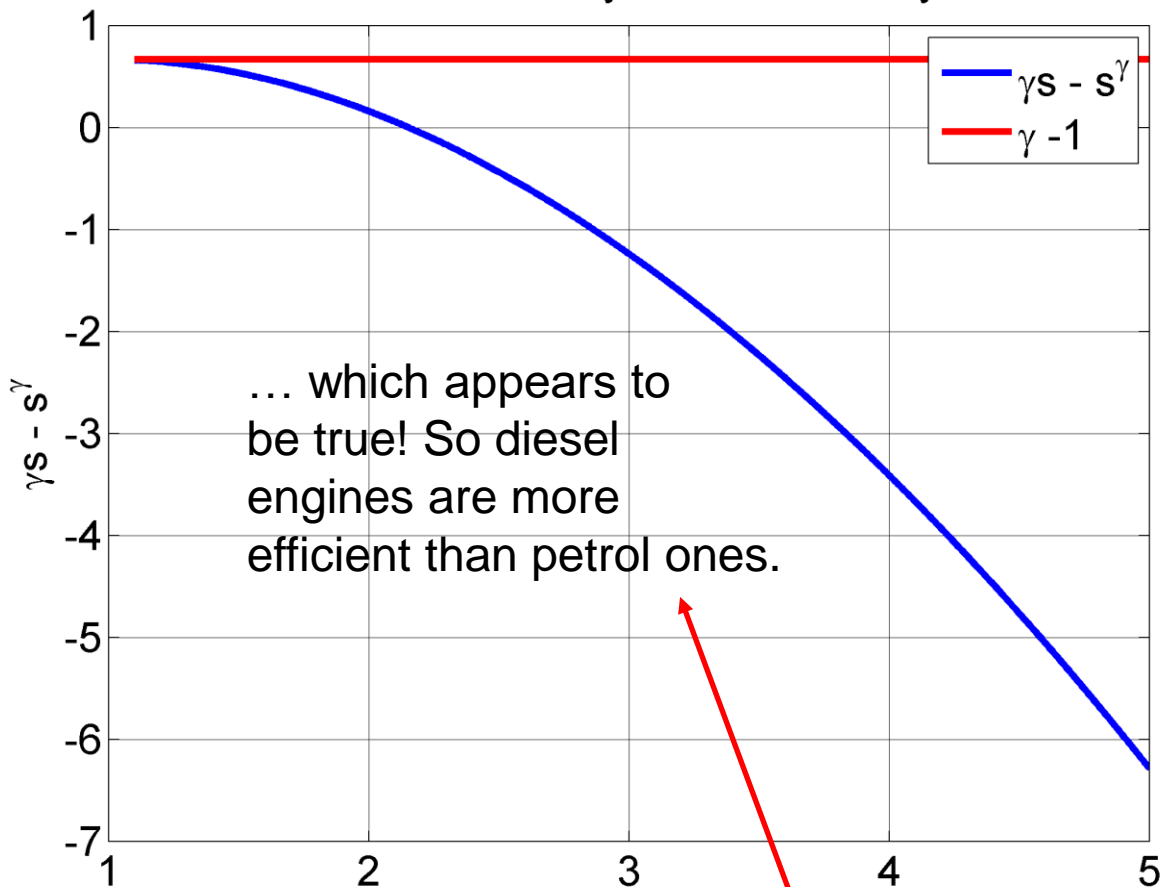
Assume about 1820 litre cylinder volume V1

Max compression ratio:
V1/V2 = 23

V3 guessed at:
V1/10.726

V or p diff	1 to 2		2 to 3		3 to 4		4 to 1	
	Adiabatic compression	Isobaric heating	Adiabatic expansion	Isochoric cooling	Adiabatic expansion	Isochoric cooling	Adiabatic expansion	Isochoric cooling
p	V	p	V	p	V	p	V	
0	1.000	1820.000	186.014	79.130	186.014	169.681	3.566	1820.000
0.01	1.016	1802.591	186.014	80.036	159.354	186.184	3.540	1820.000
0.02	1.033	1785.183	186.014	80.941	138.322	202.688	3.514	1820.000
0.03	1.050	1767.774	186.014	81.847	121.404	219.191	3.489	1820.000
0.04	1.067	1750.365	186.014	82.752	107.569	235.694	3.463	1820.000
0.05	1.085	1732.957	186.014	83.658	96.095	252.197	3.437	1820.000
0.06	1.104	1715.548	186.014	84.563	86.461	268.700	3.412	1820.000
0.07	1.122	1698.139	186.014	85.469	78.285	285.203	3.386	1820.000
0.08	1.142	1680.730	186.014	86.374	71.279	301.707	3.360	1820.000
0.09	1.162	1663.322	186.014	87.280	65.225	318.210	3.335	1820.000
0.1	1.182	1645.913	186.014	88.186	59.954	334.713	3.309	1820.000
0.11	1.204	1628.504	186.014	89.091	55.332	351.216	3.284	1820.000
0.12	1.225	1611.096	186.014	89.997	51.256	367.719	3.258	1820.000
0.13	1.248	1593.687	186.014	90.902	47.639	384.223	3.232	1820.000
0.14	1.271	1576.278	186.014	91.808	44.414	400.726	3.207	1820.000
0.15	1.295	1558.870	186.014	92.713	41.525	417.229	3.181	1820.000
0.16	1.319	1541.461	186.014	93.619	38.925	433.732	3.155	1820.000
0.17	1.344	1524.052	186.014	94.524	36.577	450.235	3.130	1820.000
0.18	1.370	1506.643	186.014	95.430	34.447	466.739	3.104	1820.000
0.19	1.397	1489.235	186.014	96.335	32.508	483.242	3.078	1820.000
0.2	1.425	1471.826	186.014	97.241	30.739	499.745	3.053	1820.000
0.21	1.453	1454.417	186.014	98.146	29.119	516.248	3.027	1820.000
0.22	1.483	1437.009	186.014	99.052	27.631	532.751	3.001	1820.000
0.23	1.513	1419.600	186.014	99.957	26.261	549.254	2.975	1820.000
0.24	1.543	1402.191	186.014	100.862	25.000	565.757	2.949	1820.000
0.25	1.573	1384.782	186.014	101.767	23.848	582.260	2.923	1820.000
0.26	1.603	1367.373	186.014	102.672	22.806	598.763	2.897	1820.000
0.27	1.633	1349.964	186.014	103.577	21.874	615.266	2.871	1820.000
0.28	1.663	1332.555	186.014	104.482	21.052	631.769	2.845	1820.000
0.29	1.693	1315.146	186.014	105.387	20.340	648.272	2.819	1820.000
0.3	1.723	1297.737	186.014	106.292	19.738	664.775	2.793	1820.000
0.31	1.753	1280.328	186.014	107.197	19.246	681.278	2.767	1820.000
0.32	1.783	1262.919	186.014	108.102	18.864	697.781	2.741	1820.000
0.33	1.813	1245.510	186.014	109.007	18.592	714.284	2.715	1820.000
0.34	1.843	1228.101	186.014	109.912	18.430	730.787	2.689	1820.000
0.35	1.873	1210.692	186.014	110.817	18.378	747.290	2.663	1820.000
0.36	1.903	1193.283	186.014	111.722	18.436	763.793	2.637	1820.000
0.37	1.933	1175.874	186.014	112.627	18.604	780.296	2.611	1820.000
0.38	1.963	1158.465	186.014	113.532	18.882	796.799	2.585	1820.000
0.39	1.993	1141.056	186.014	114.437	19.270	813.302	2.559	1820.000
0.4	2.023	1123.647	186.014	115.342	19.768	829.805	2.533	1820.000
0.41	2.053	1106.238	186.014	116.247	20.376	846.308	2.507	1820.000
0.42	2.083	1088.829	186.014	117.152	21.094	862.811	2.481	1820.000
0.43	2.113	1071.420	186.014	118.057	21.922	879.314	2.455	1820.000
0.44	2.143	1054.011	186.014	118.962	22.860	895.817	2.429	1820.000
0.45	2.173	1036.602	186.014	119.867	23.908	912.320	2.403	1820.000
0.46	2.203	1019.193	186.014	120.772	25.066	928.823	2.377	1820.000
0.47	2.233	1001.784	186.014	121.677	26.334	945.326	2.351	1820.000
0.48	2.263	984.375	186.014	122.582	27.712	961.829	2.325	1820.000
0.49	2.293	966.966	186.014	123.487	29.200	978.332	2.299	1820.000

Diesel efficiency > Otto efficiency



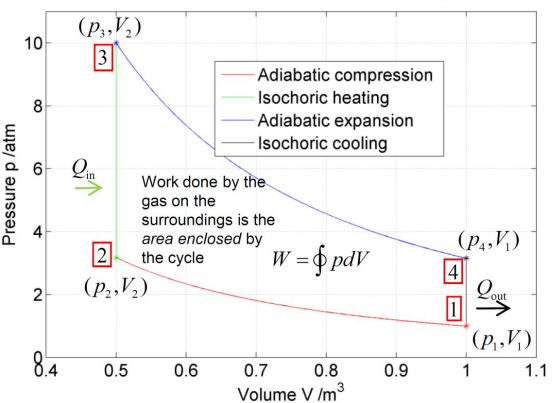
$$r = \frac{V_1}{V_2} \quad s = \frac{V_3}{V_2}$$

$$\eta = \frac{W}{Q_{in}}$$

$$\eta_{diesel} = 1 - \frac{1}{r^{\gamma-1}} \left(\frac{s^{\gamma} - 1}{\gamma(s-1)} \right)$$

$$\eta_{otto} = 1 - \frac{1}{r^{\gamma-1}}$$

Otto cycle
 $T_1 = 20^{\circ}\text{C}, T_2 = 192.1085^{\circ}\text{C}, T_3 = 1192^{\circ}\text{C}, T_4 = 649.8922^{\circ}\text{C}$
 $W = 191.9293\text{kJ}, Q_{in} = 518.6724\text{kJ}, \text{Efficiency} = W/Q_{in} = 37.0039\%$

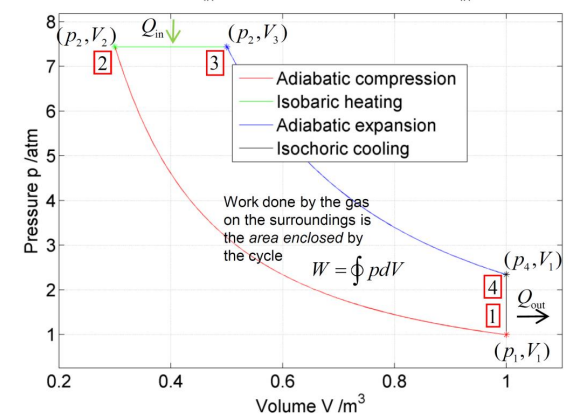


$$s = \frac{V_3}{V_2}$$

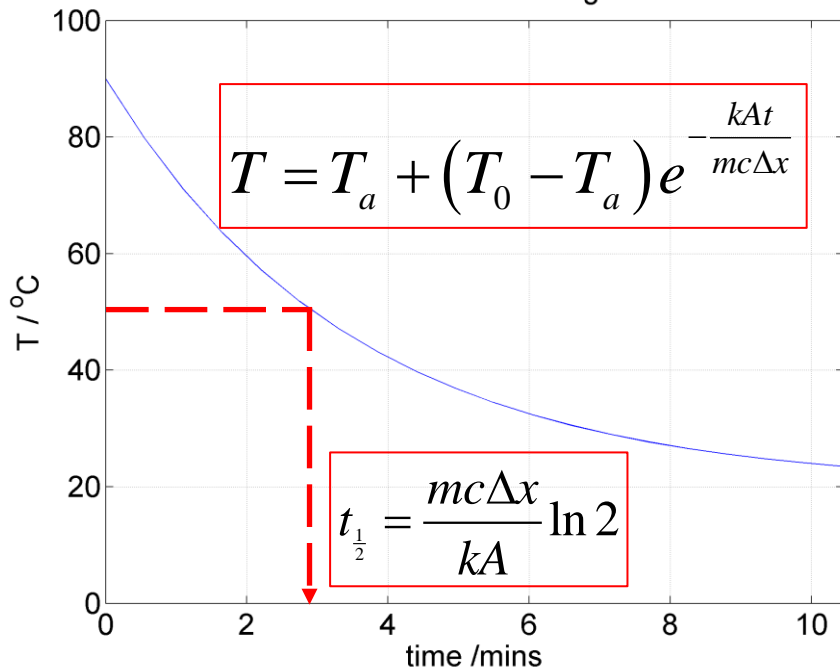
$$\eta_{diesel} > \eta_{otto}$$

$$\Rightarrow \gamma - 1 > \gamma s - s^{\gamma}$$

Diesel cycle
 $T_1 = 20^{\circ}\text{C}, T_2 = 380.8128^{\circ}\text{C}, T_3 = 816.6881^{\circ}\text{C}, T_4 = 413.4605^{\circ}\text{C}$
 $W = 172.7357\text{kJ}, Q_{in} = 376.835\text{kJ}, \text{Efficiency} = W/Q_{in} = 45.8386\%$



Newtonian Cooling



If the specific heat capacity of the fluid is c , and the vessel contains m kg of fluid

$$dQ = -mcdT$$

If we assume the heat capacity is independent of temperature

$$\frac{dT}{dt} = -\frac{kA}{mc\Delta x} (T - T_a)$$

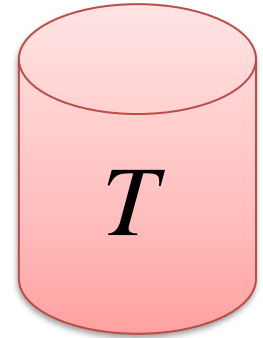
$$\int_{T_0}^T \frac{dT}{T - T_a} = -\frac{kA}{mc\Delta x} \int_0^t dt$$

$$\left[\ln|T - T_a| \right]_{T_0}^T = -\frac{kAt}{mc\Delta x}$$

Q is the heat transferred from the vessel to the surroundings
 k is the thermal conductivity of the vessel
 Δx is the thickness of the vessel
 A is the surface area of the vessel

$$\frac{dQ}{dt} = kA \frac{(T - T_a)}{\Delta x}$$

T_a T



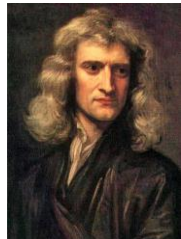
Joseph Fourier
(1768-1830)

“Heat flow via conduction is proportional to temperature gradient”

$$\ln \left(\frac{T - T_a}{T_0 - T_a} \right) = -\frac{kAt}{mc\Delta x}$$

$$\frac{T - T_a}{T_0 - T_a} = e^{-\frac{kAt}{mc\Delta x}}$$

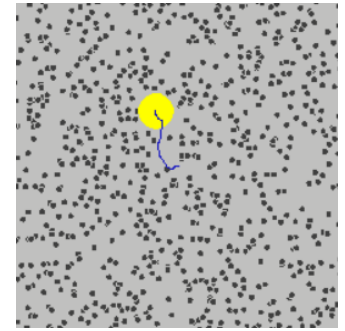
$$T = T_a + (T_0 - T_a) e^{-\frac{kAt}{mc\Delta x}}$$



Isaac Newton
(1643-1727)

Brownian motion – a random walk

Brownian motion, initially observed as the random jittering of pollen grains in a microscope slide, is due to the random jostling of molecular motion. In the base of the pollen grains, it is the smaller (invisible) air molecules which are colliding at random. How far will a given particle move in a specified time, given its motion is random?



[Brownian motion simulation](#)

Consider motion in one direction in N steps of fixed length l . The caveat is that each step is either forward or backwards, and the direction is 'chosen' randomly.

The total displacement is $x = l \sum_{i=1}^N a_i$ where $a_i = -1$ or 1

A sensible measure of the distance travelled is the *root-mean-square (RMS) displacement*:

$$\sqrt{\langle x^2 \rangle} = l \sqrt{\left\langle \left(\sum_{i=1}^N a_i \right)^2 \right\rangle} = l \sqrt{\left\langle \sum_{i=1}^N a_i^2 + \sum_{i=1, i \neq j}^N \sum_{j=1}^N a_i a_j \right\rangle}$$

$$\left\langle \sum_{i=1}^N a_i^2 \right\rangle = N \quad \text{and} \quad \left\langle \sum_{i=1, i \neq j}^N \sum_{j=1}^N a_i a_j \right\rangle = 0 \quad \leftarrow \text{Since } a \text{ is a random choice between } -1 \text{ and } 1$$

$$\therefore \sqrt{\langle x^2 \rangle} = l \sqrt{N}$$



Robert Brown
(1773-1858)

$$\sqrt{\langle x^2 \rangle} = l \sqrt{\left\langle \left(\sum_{i=1}^N a_i \right)^2 \right\rangle} = l \sqrt{\left\langle \sum_{i=1}^N a_i^2 + \sum_{i=1, i \neq j}^N \sum_{j=1}^N a_i a_j \right\rangle}$$

$$\left\langle \sum_{i=1}^N a_i^2 \right\rangle = N \quad \text{and} \quad \left\langle \sum_{i=1, i \neq j}^N \sum_{j=1}^N a_i a_j \right\rangle = 0$$

$$\therefore \sqrt{\langle x^2 \rangle} = l \sqrt{N}$$

$$x = l \sum_{i=1}^N a_i$$

$$a_i = -1 \text{ or } 1$$

1D random walk

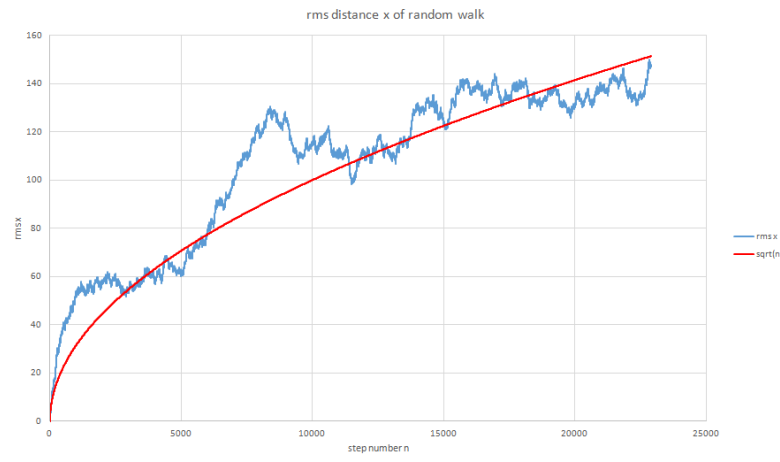
$\sqrt{\langle v^2 \rangle}$ We might use the RMS speed here

If the average molecular speed is $\langle v \rangle$, the number of steps in t seconds is: $N = \frac{\langle v \rangle t}{l}$

Hence the RMS random walk displacement in t seconds is predicted to be:

$$\sqrt{\langle x^2 \rangle} = l \sqrt{N} = \sqrt{l \langle v \rangle t}$$

The step size l can be associated with the **mean free path** between molecular collisions. We can define the mean free path to be the average distance travelled by a molecule in time t divided by the number of molecules it will likely collide with in that time.



$$l = \frac{\sqrt{\langle v^2 \rangle} t}{\underbrace{\pi d^2 \sqrt{2} \sqrt{\langle v^2 \rangle} t}_{\text{'Interaction volume'}} \times n}$$

Distance travelled by molecule

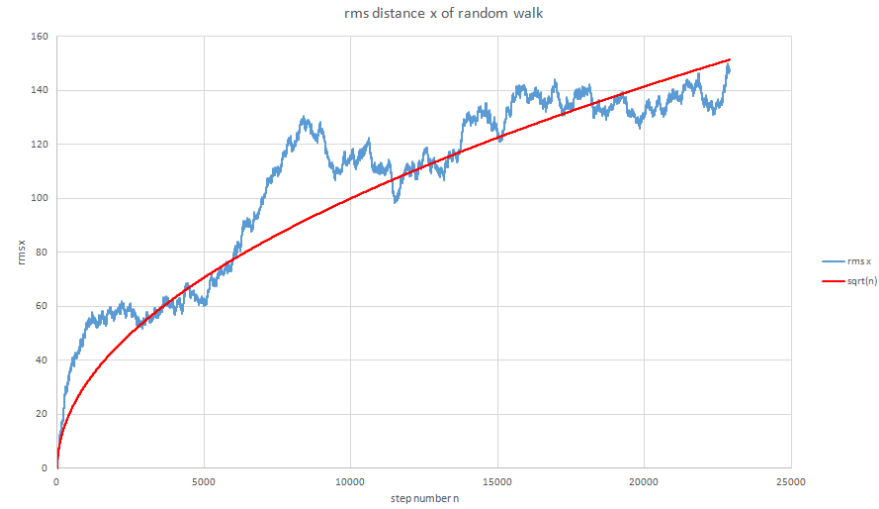
'Interaction volume'

number of molecules per unit volume

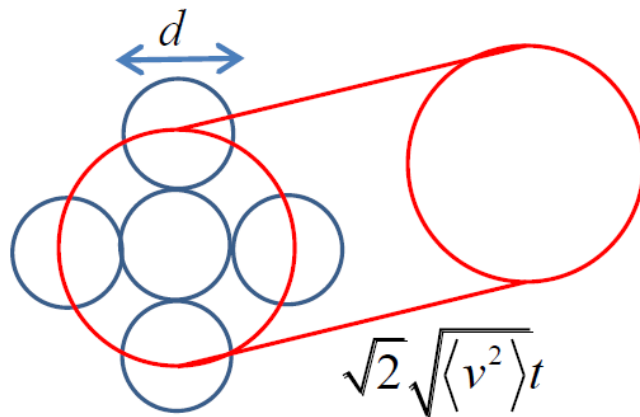
$$l = \frac{1}{\pi \sqrt{2} d^2 n}$$

Mean free path

The interaction volume is root 2 larger because all molecules are in *relative* motion. Hence the length of the 'interaction tube' is proportional to the average *relative* speed



Average distance between collisions



velocities of molecules *i* and *j*

$$\langle v_{rel} \rangle = \sqrt{\langle |\mathbf{v}_i - \mathbf{v}_j|^2 \rangle}$$

$$\langle v_{rel} \rangle = \sqrt{\langle (v_i^2 + v_j^2 - 2\mathbf{v}_i \cdot \mathbf{v}_j) \rangle}$$

$$\langle v_{rel} \rangle = \sqrt{2\langle v^2 \rangle - 2\langle \mathbf{v}_i \cdot \mathbf{v}_j \rangle}$$

$$\langle v_{rel} \rangle = \sqrt{2} \sqrt{\langle v^2 \rangle}$$

Mean free path

Colliding particles, assumed to be circular with diameter *d*

We can determine the **mean free path** for an ideal gas by using the **Ideal Gas Equation**

Since the number of moles is $\frac{nV}{N_A}$ ← volume

pressure → $pV = \frac{nV}{N_A} RT$ ← Absolute temperature

$\therefore n = \frac{p}{k_B T}$ ← Molar gas constant
 $R = 8.314 \text{ Jmol}^{-1} \text{ K}^{-1}$

← Boltzmann's constant $k_B = 1.38 \times 10^{-23} \text{ JK}^{-1}$

Hence

$$l = \frac{k_B T}{\pi \sqrt{2} d^2 p}$$

If we divide this by the particle diameter d we arrive at **Knudsen's number (Kn)**. This dimensionless constant determines whether our statistical mechanics argument is valid, or whether a 'continuum' concept is needed.

The latter model is what is used to describe much of **fluid mechanics** i.e. where we consider the fluid as a continuously varying entity rather than a series of discrete, and randomly moving, molecules colliding.

$$l = \frac{k_B T}{\pi \sqrt{2} d^2 p}$$

Kn \ll 1 Continuum
 Kn > 1 Statistical mechanics

For a typical air molecule on Earth

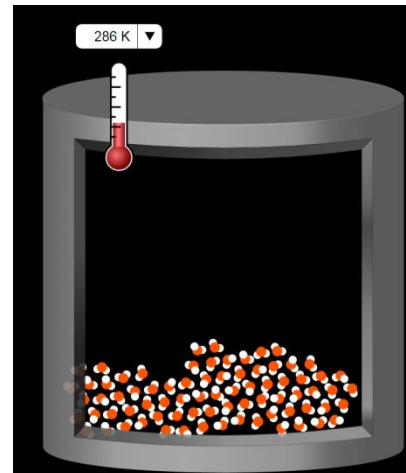
$$d = 0.3 \text{ nm}, \quad p = 10^5 \text{ Pa}, \quad T = 293 \text{ K}$$

$$\therefore l = 1.0 \times 10^{-7} \text{ m}$$

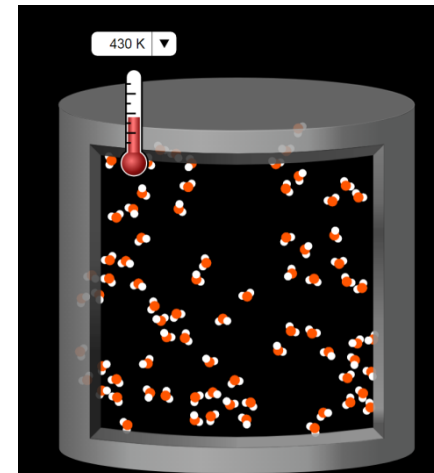
$$\therefore \text{Kn} = \frac{l}{d} = \boxed{333}$$

So a statistical argument is justified

Fluid

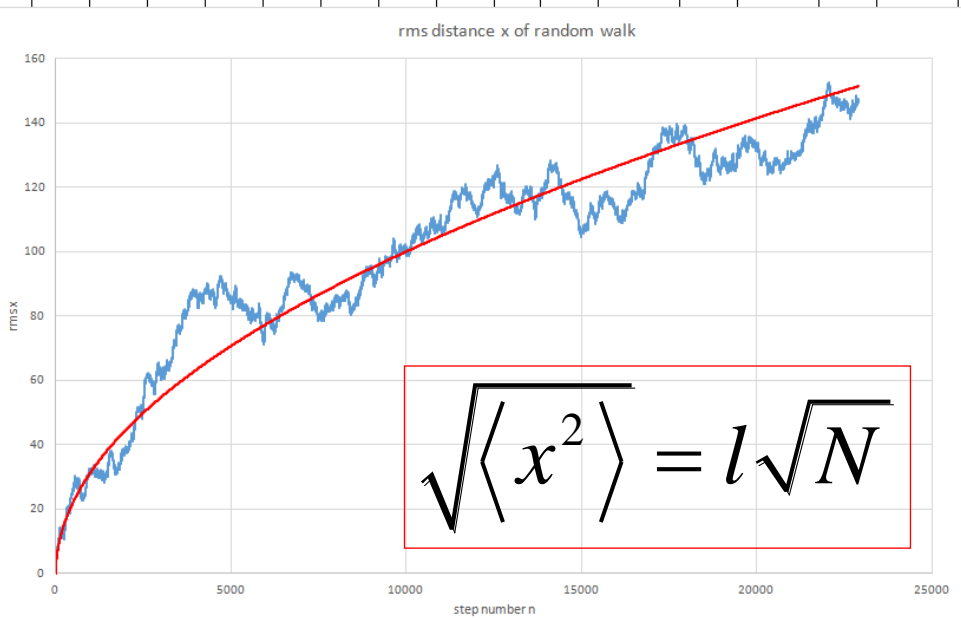


Gas



STEP SIZE = 1

n	sqrt(n)	x	x	x	x	x	x	x	x	x	x	x	x	x^2	x^2	x^2	x^2	x^2	x^2	x^2	x^2	x^2	x^2	x^2	rms x
0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
1	1	1	-1	1	1	1	-1	1	1	-1	-1	1	-1	1	1	1	1	1	1	1	1	1	1	1	1
2	1.4142	2	-2	2	2	2	0	0	2	-2	0	0	0	4	4	4	4	4	0	0	4	4	0	0	1.528
3	1.7321	3	-3	1	3	3	3	3	3	3	3	3	3	9	9	9	9	9	9	9	9	9	9	9	1.915
4	2	4	-4	2	4	4	4	4	4	4	4	4	4	16	16	16	16	16	16	16	16	16	16	16	2.38
5	2.2361	5	-5	1	5	5	5	5	5	5	5	5	5	25	25	25	25	25	25	25	25	25	25	25	2.887
6	2.4495	4	-4	0	4	4	4	4	4	4	4	4	4	16	16	16	16	16	16	16	16	16	16	16	2.708
7	2.6458	3	-5	-1	5	5	5	5	5	5	5	5	5	25	25	25	25	25	25	25	25	25	25	25	2.887
8	2.8284	4	-4	-2	6	6	6	6	6	6	6	6	6	36	36	36	36	36	36	36	36	36	36	36	3
9	3	3	-5	-3	5	5	5	5	5	5	5	5	5	25	25	25	25	25	25	25	25	25	25	25	3.109
10	3.1623	2	-6	-2	4	4	4	4	4	4	4	4	4	16	16	16	16	16	16	16	16	16	16	16	2.944
11	3.3166	1	-7	-1	5	5	5	5	5	5	5	5	5	25	25	25	25	25	25	25	25	25	25	25	3.512
12	3.4641	0	-8	0	6	6	6	6	6	6	6	6	6	36	36	36	36	36	36	36	36	36	36	36	4
13	3.6056	-1	-7	-1	7	7	7	7	7	7	7	7	7	49	49	49	49	49	49	49	49	49	49	49	4.359
14	3.7417	0	-6	-2	8	8	8	8	8	8	8	8	8	64	64	64	64	64	64	64	64	64	64	64	4.546
15	3.873	-1	-5	-1	9	9	9	9	9	9	9	9	9	81	81	81	81	81	81	81	81	81	81	81	4.865
16	4	-2	-4	0	8	8	8	8	8	8	8	8	8	64	64	64	64	64	64	64	64	64	64	64	4.655
17	4.1231	-3	-3	-1	7	7	7	7	7	7	7	7	7	49	49	49	49	49	49	49	49	49	49	49	4.655
18	4.2426	-4	-2	-2	8	8	8	8	8	8	8	8	8	64	64	64	64	64	64	64	64	64	64	64	5.196
19	4.3589	-3	-3	-1	7	7	7	7	7	7	7	7	7	49	49	49	49	49	49	49	49	49	49	49	5.066
20	4.4721	-2	-2	-2	8	8	8	8	8	8	8	8	8	64	64	64	64	64	64	64	64	64	64	64	5.715
21	4.5826	-1	-3	-3	7	7	7	7	7	7	7	7	7	49	49	49	49	49	49	49	49	49	49	49	5.627
22	4.6904	-2	-4	-4	8	8	8	8	8	8	8	8	8	64	64	64	64	64	64	64	64	64	64	64	5.538
23	4.7958	-1	-5	-3	9	9	9	9	9	9	9	9	9	81	81	81	81	81	81	81	81	81	81	81	6.137
24	4.899	-2	-4	-2	8	8	8	8	8	8	8	8	8	64	64	64	64	64	64	64	64	64	64	64	1
25	5	-3	-5	-1	7	9	9	9	9	9	9	9	9	9	25	1	49	81	9	121	9	9	9	9	9
26	5.099	-2	-4	-2	8	8	8	8	8	8	8	8	8	4	16	4	64	64	16	100	16	1	1	1	1
27	5.1962	-3	-5	-1	7	7	7	7	7	7	7	7	7	9	25	1	49	49	9	121	9	2	2	2	2
28	5.2915	-2	-6	0	6	6	6	6	6	6	6	6	6	4	36	0	36	36	4	144	16	1	1	1	1
29	5.3852	-1	-5	1	7	5	5	5	5	5	5	5	5	1	25	1	49	25	1	121	25	9	9	9	9
30	5.4772	-2	-4	0	8	4	0	10	-6	2	-10	4	-2	4	16	0	64	16	0	100	36	4	4	4	4
31	5.5678	-3	-5	1	9	3	1	11	-5	3	-9	5	-1	9	25	1	81	9	1	121	25	9	9	9	9



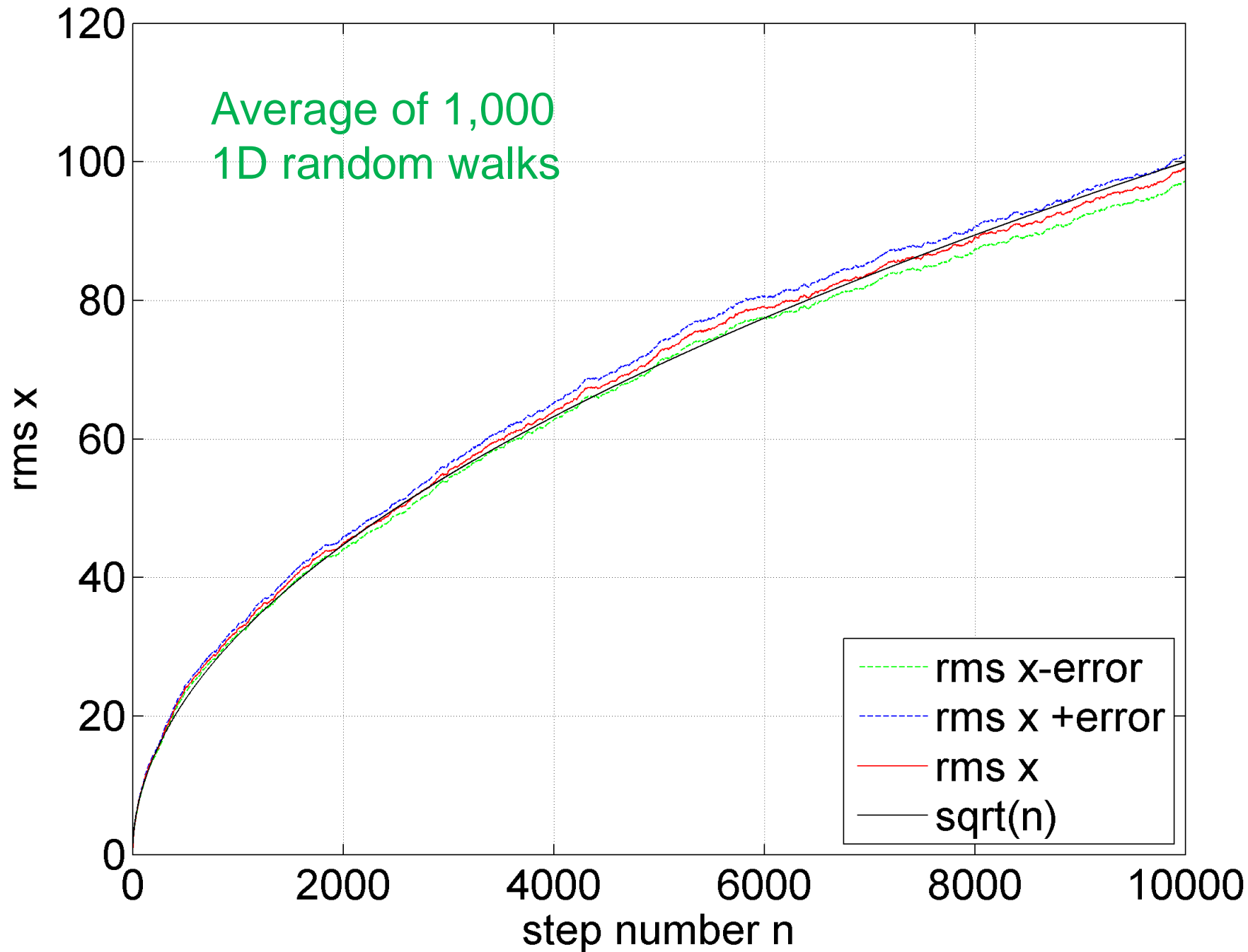
$$x = l \sum_{i=1}^N a_i$$

$$a_i = -1 \text{ or } 1$$

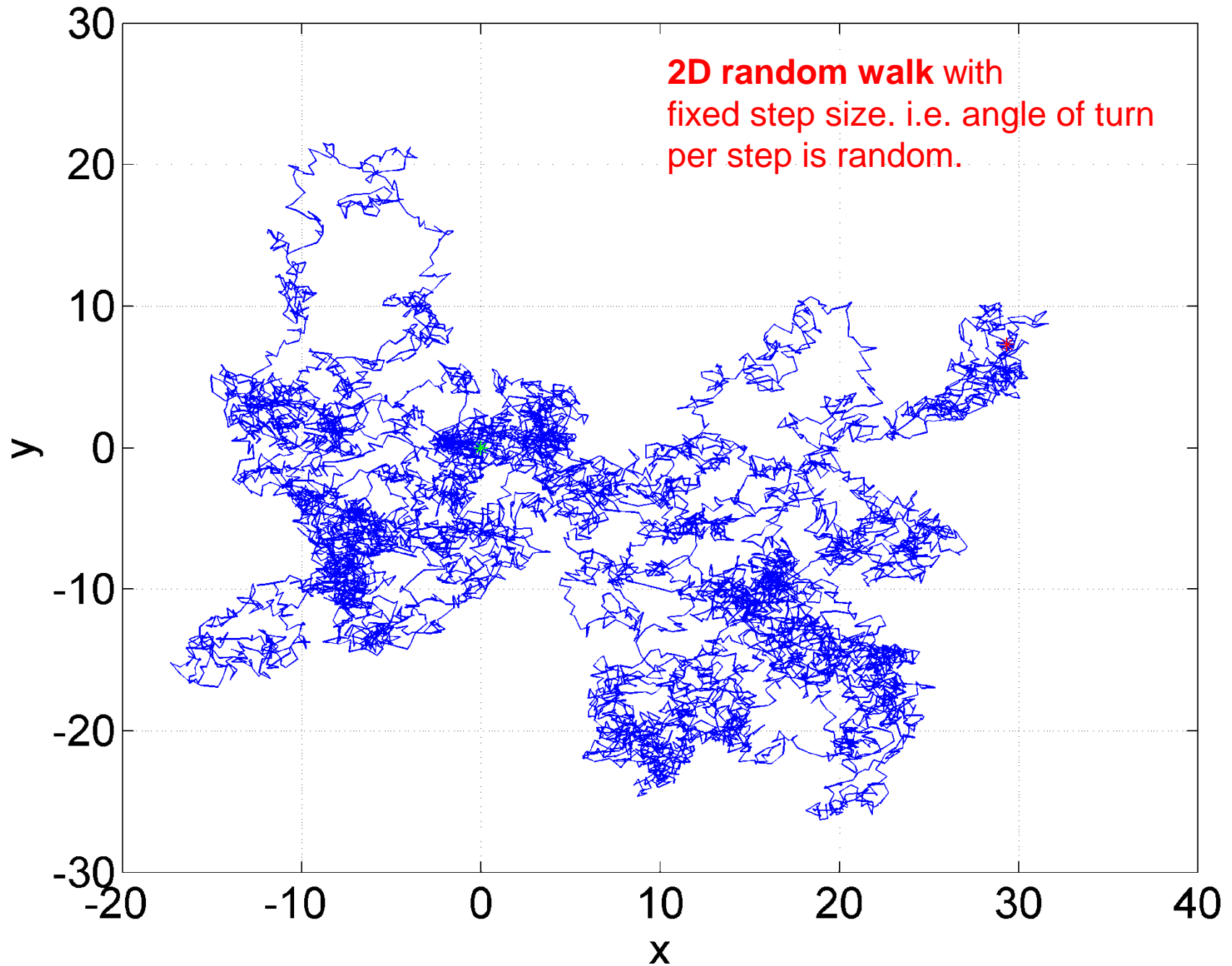
$$l = 1$$

1D random walk simulation in Excel

Random walk $l=1000$, $N=10000$



Random walk. Max step size = 1, N = 10000

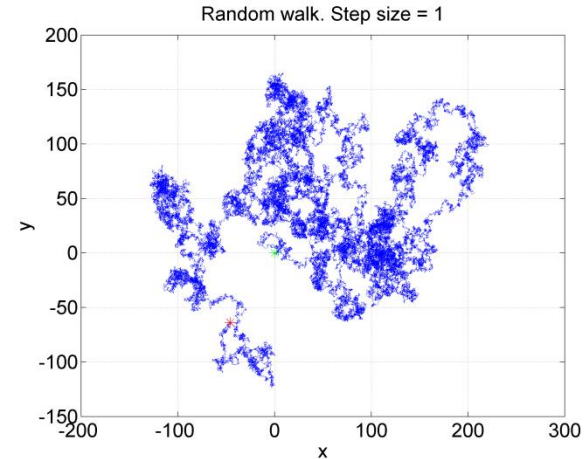


```

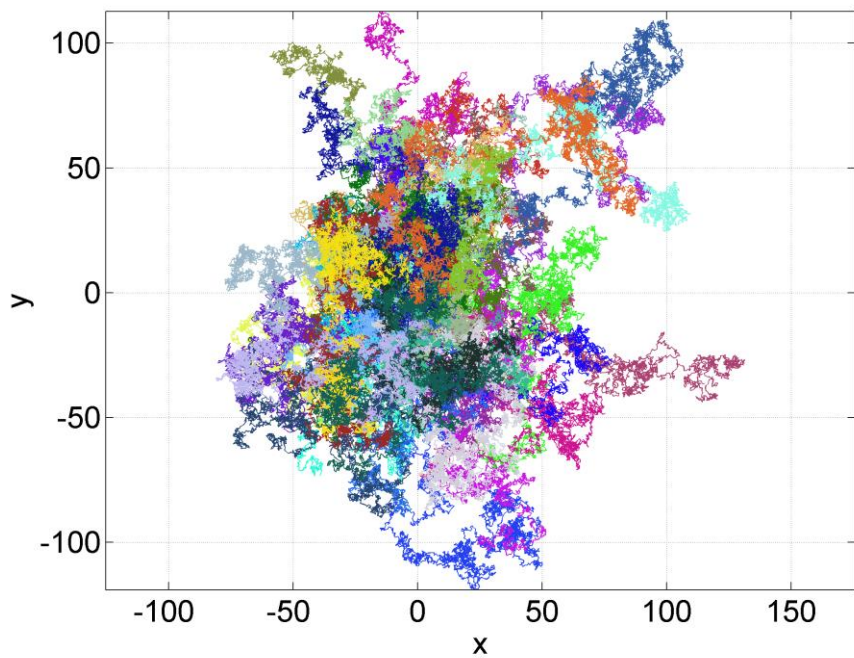
1 | % A visual representation of a random walk.
2 | % Step sizes are fixed, but directions are random.
3 |
4 | %Number of steps
5 | N = 1e6;
6 |
7 | %Fixed step size
8 | s = 1;
9 |
10 | %Initilize x,y position vectors, starting from the origin.
11 | x = zeros(1,N); y = zeros(1,N);
12 |
13 | %Determine random walk
14 | for n=2:N
15 |     theta = 2*pi*rand;
16 |     x(n) = x(n-1) + s*cos(theta); y(n) = y(n-1) + s*sin(theta);
17 | end
18 |
19 | %Plot random walk
20 | plot(x,y,'b-'); hold on;
21 | plot( x(1),y(1),'g*' ); plot( x(end),y(end),'r*' );
22 | xlabel('x'); ylabel('y'); title( ['Random walk. Step size = ',num2str(s)] );
23 | grid on;
24 |
25 | %Print a PNG file of the random walk
26 | print( gcf, 'random walk.png', '-dpng', '-r300' );
27 |
28 | %End of code

```

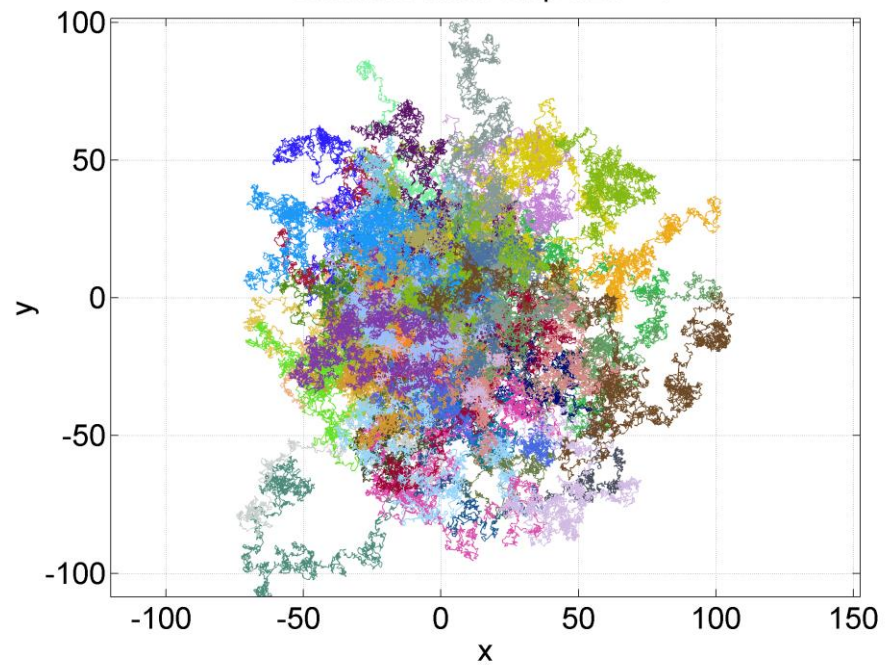
2D random walk MATLAB simulation



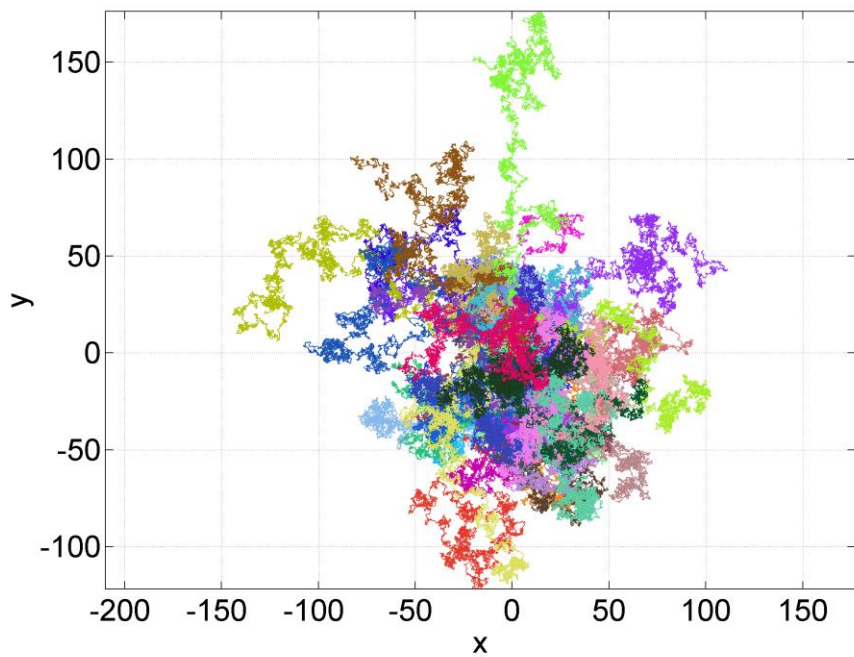
Random walk. Step size = 1



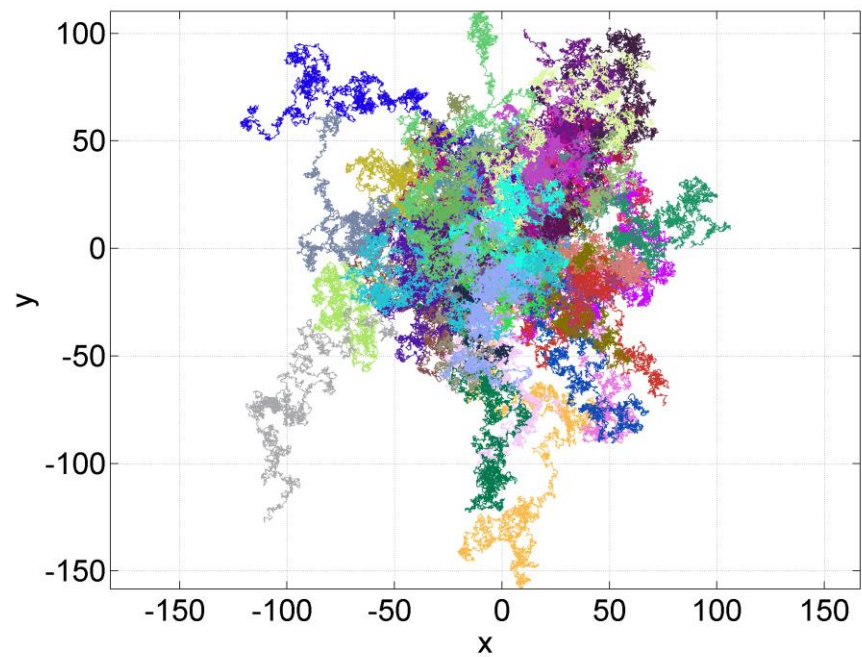
Random walk. Step size = 1



Random walk. Step size = 1



Random walk. Step size = 1



```
7 function random_walks
```

```
8  
9 P = 42; %Numbers of random walks  
10 N = 5000; %Number of steps  
11 s = 1; %Fixed step size  
12 fsize = 18; %Graph fontsize
```

```
13  
14 %Initialize axes and then plot random walks
```

```
15 axes('nextplot','add','fontsize',fsize);
```

```
16 for n=1:P
```

```
17     [x,y] = randomwalk(N,s);
```

```
18     RGB = rand(1,3); plot(x,y,'-','color',RGB);
```

```
19 end
```

```
20 xlabel('x'); ylabel('y'); title( ['Random walk. Step size = ',num2str(s)] );
```

```
21 grid on; axis equal; box on;
```

```
22  
23 %Print a PNG file of the random walk
```

```
24 filename = ['random walks ',strrep(datestr(now),':','-'),' .png'];
```

```
25 print( gcf, filename, '-dpng', '-r300' );
```

```
26 close(gcf);
```

```
27  
28 %%
```

```
29  
30 %Random walk generator
```

```
31 function [x,y] = randomwalk(N,s)
```

```
32 x = zeros(1,N); y = zeros(1,N);
```

```
33 for n=2:N
```

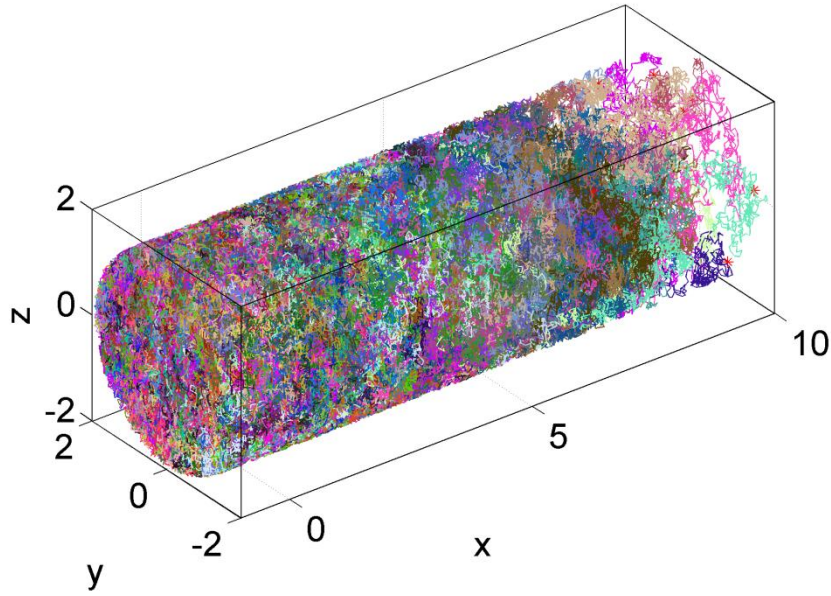
```
34     theta = 2*pi*rand;
```

```
35     x(n) = x(n-1) + s*cos(theta); y(n) = y(n-1) + s*sin(theta);
```

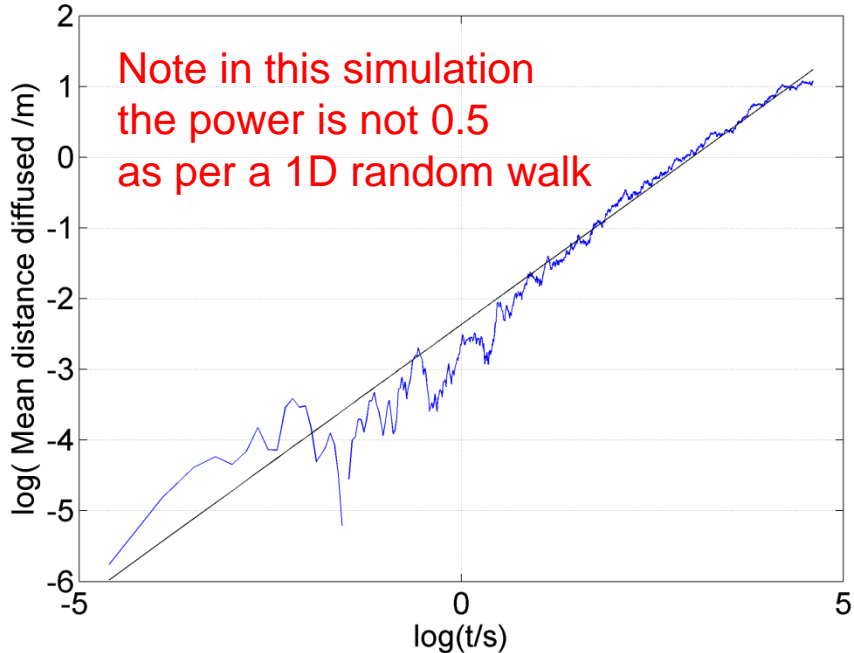
```
36 end
```

MATLAB implementation
of multiple random
walks (in a loop)

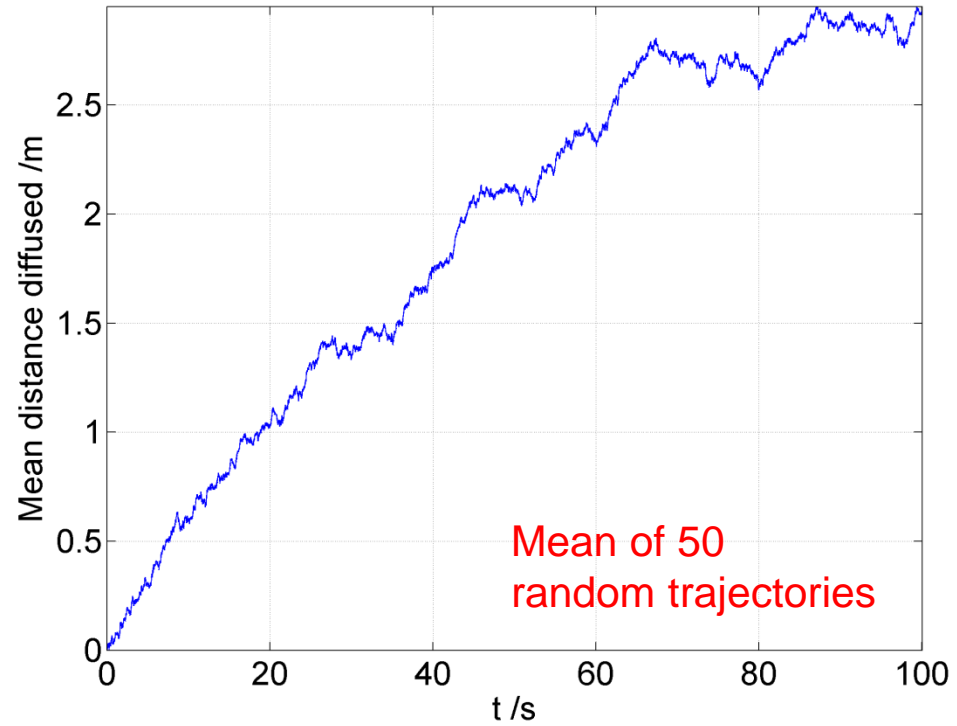
Diffusion. Particle speed =10m/s



Mean x vs t for diffusion, v = 10m/s
 $x = 0.0937t^{0.784}$



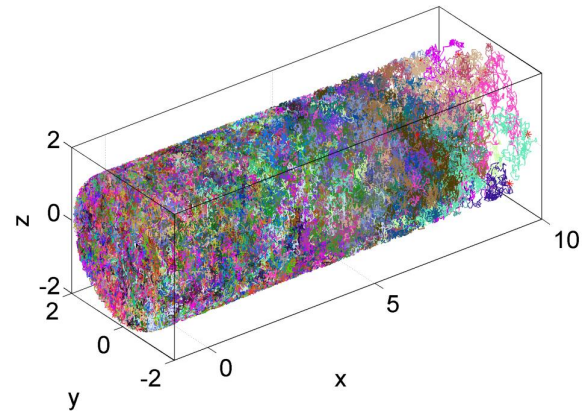
x vs t for diffusion, v = 10m/s



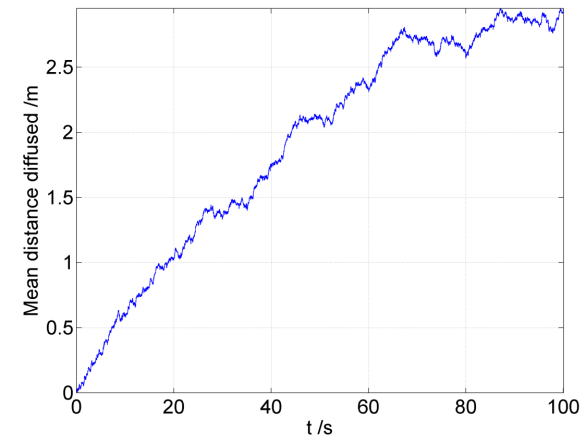
We can compute a 3D **diffusion** model *efficiently* by using a random walk.

The random walk gets around the need to keep track of thousands of particles and their collisions.

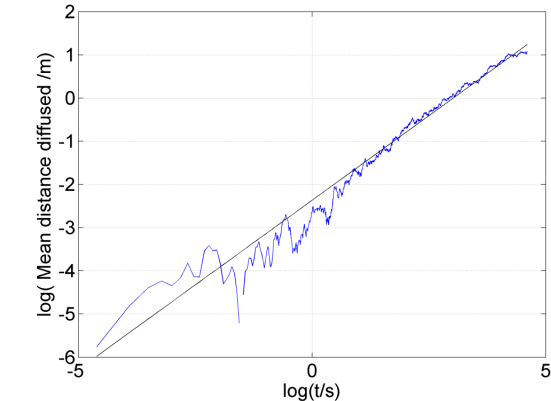
Diffusion. Particle speed =10m/s



x vs t for diffusion, v = 10m/s



Mean x vs t for diffusion, v = 10m/s
 $x = 0.0937t^{0.784}$



`%Determine trajectory`

```
function [x,y,z,t] = diffusion_trajectory(a,r,s,dt,tmax)
```

`%Initialize x,y,z,t coordinates`

```
x = 0; y = 0; z = 0; t = 0;
```

`%Compute the random walk!`

```
n=1;
```

```
while t(end) < tmax
```

`%Choose random direction angles`

```
azi = 2*pi*rand; elev = -0.5*pi + pi*rand;
```

`%Define magnitude of displacement during timestep`

```
d = s*dt;
```

`%Compute next x,y position`

```
dx = d*cos(elev)*cos(azi);
```

```
dy = d*cos(elev)*sin(azi);
```

```
dz = d*sin(elev);
```

`%Check if particle has hit a side wall`

```
if ( ( y(end)+dy )^2 + ( z(end) + dz )^2 ) > r^2
```

```
    dz = -dz; dy = -dy;
```

```
end
```

`%Check if particle has hit back wall`

```
if ( x(end) + dx ) < a
```

```
    dx = -dx;
```

```
end
```

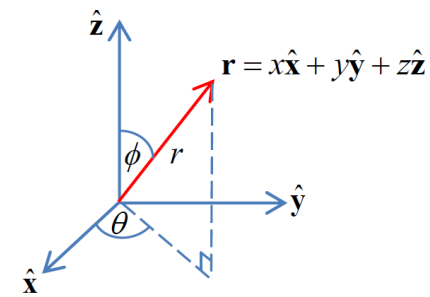
`%Update x,y,z trajectory`

```
x(n+1) = x(n) + dx; y(n+1) = y(n) + dy; z(n+1) = z(n) + dz;
```

```
t(n+1) = t(n) + dt; n = n+1;
```

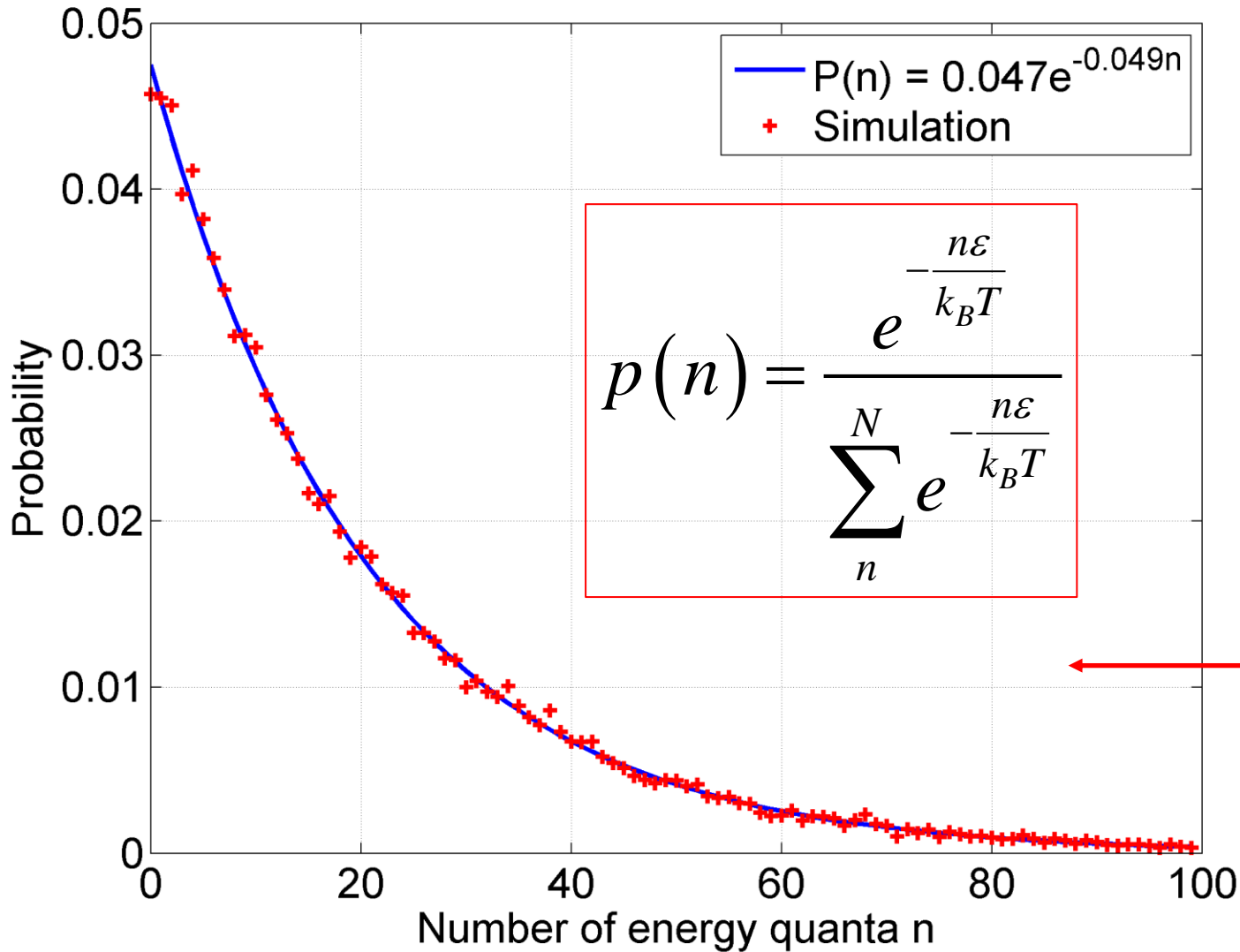
```
end
```

Random angles
in azimuth and elevation



Assume elastic
collisions with walls

Boltzmann distribution simulation. $I=100$, $N=10000$, $M=500$



$N = 10,000$
energy quanta
shared randomly
among $M = 500$
molecules

Process
repeated 100
times

Normalized
histogram



$$k_B = 1.38 \times 10^{-23} \text{ JK}^{-1}$$

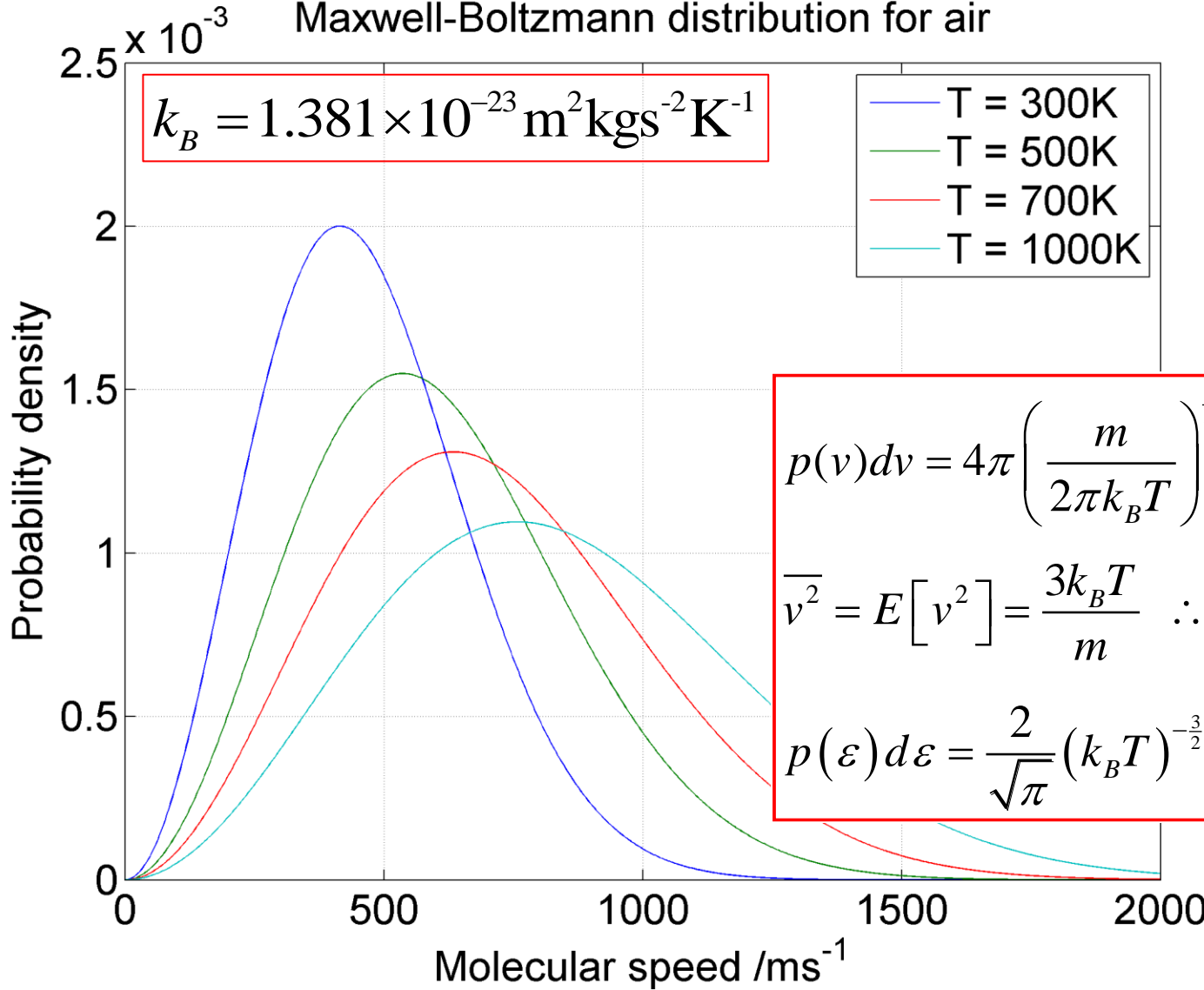


Ludwig Boltzmann
1844-1906

Maxwell-Boltzmann distribution for air



James Clerk Maxwell
(1831–1879)



$$p(v)dv = 4\pi \left(\frac{m}{2\pi k_B T} \right)^{\frac{3}{2}} v^2 e^{-\frac{1}{2} \frac{mv^2}{k_B T}} dv$$

$$\overline{v^2} = E[v^2] = \frac{3k_B T}{m} \quad \therefore v_{rms} = \sqrt{\overline{v^2}} = \sqrt{\frac{3k_B T}{m}}$$

$$p(\varepsilon)d\varepsilon = \frac{2}{\sqrt{\pi}} (k_B T)^{-\frac{3}{2}} \sqrt{\varepsilon} e^{-\frac{\varepsilon}{k_B T}} d\varepsilon$$

$$E \left[\frac{1}{2} mv^2 \right] = \frac{3}{2} k_B T$$

i.e. absolute temperature is proportional to mean KE of molecules



Ludwig Boltzmann
1844-1906