

SHM differential equation: $\ddot{x} = -\omega^2 x$ with solution $x(t) = A \cos(\omega t - \phi)$. i.e. an oscillation of frequency $f = \frac{1}{2\pi} \omega$ and period $T = 1/f$. i.e. $\omega = 2\pi/T$. A is the amplitude and ϕ is a constant phase shift. Note $\ddot{x} = d^2x/dt^2$, $\dot{x} = dx/dt$.

Also: $\ddot{x} = \dot{x} \frac{d\dot{x}}{dx}$. So $\dot{x} \frac{d\dot{x}}{dx} = -\omega^2 x \Rightarrow \frac{1}{2} \dot{x}^2 = -\frac{1}{2} \omega^2 x^2 + c \Rightarrow \dot{x}^2 = \omega^2 (A^2 - x^2)$. i.e. $\dot{x} = \pm \omega \sqrt{A^2 - x^2}$

Note also that: $\dot{x}(t) = -\omega A \sin(\omega t - \phi)$ and $A^2 \sin^2(\omega t - \phi) = A^2 - A^2 \cos^2(\omega t - \phi) = A^2 - x^2$.

Example: mass on a spring (where x is the displacement from equilibrium)

Newton II: $m\ddot{x} = -kx \therefore \ddot{x} = -\frac{k}{m}x$. So SHM with $\omega = \sqrt{\frac{k}{m}}$.

Energy conservation: $\frac{1}{2}m\dot{x}^2 + \frac{1}{2}kx^2 = \frac{1}{2}kA^2 \Rightarrow m\dot{x}\ddot{x} + kx\dot{x} = 0 \Rightarrow \dot{x} \left(\ddot{x} + \frac{k}{m}x \right) = 0 \therefore \ddot{x} = -\frac{k}{m}x$.

Damped SHM: $\ddot{x} + 2\gamma\dot{x} + \omega^2 x = 0$. e.g. mass on a spring with damping force proportional to velocity, but acts in opposition to movement. Newton II: $m\ddot{x} = -kx - 2\gamma m\dot{x}$. Note each has *two* constants to be found from $x(0)$ and $\dot{x}(0)$.

If $\omega > \gamma$: 'underdamped' and solutions are oscillatory.

$$x(t) = A_1 e^{-\gamma t} \cos\left(t\sqrt{\omega^2 - \gamma^2} - \Phi\right)$$

If $\omega < \gamma$: 'overdamped' and solutions don't oscillate.

$$x(t) = e^{-\gamma t} \left(A_1 e^{t\sqrt{\gamma^2 - \omega^2}} + A_2 e^{-t\sqrt{\gamma^2 - \omega^2}} \right)$$

If $\omega = \gamma$: 'critically damped' and solutions don't quite oscillate.

$$x(t) = e^{-\gamma t} (A_1 + A_2 t)$$

The time variation of these exponentially decaying terms is known as the *transient*, or 'impulse response.'

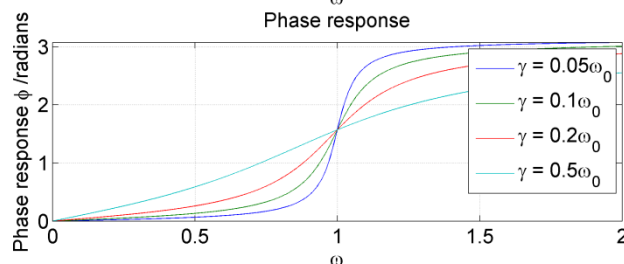
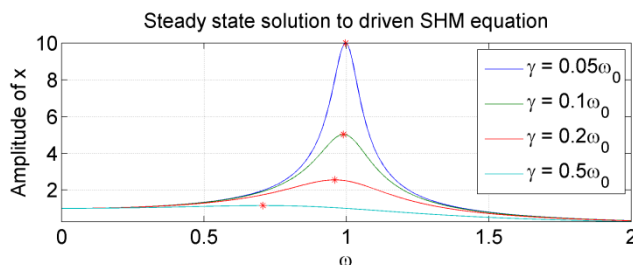
Driven SHM and resonance: $\ddot{x} + 2\gamma\dot{x} + \omega_0^2 x = A_0 \omega_0^2 \sin \omega t$.

If *underdamped*, **steady state solution** (i.e. when the transient decays sufficiently to be neglected i.e. when $\gamma t \gg 1$) is:

$$x(t) = A \sin(\omega t - \phi). \quad A(\omega) = A_0 \omega_0^2 / \sqrt{(\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2}. \quad \phi = \tan^{-1} \left(\frac{2\gamma\omega}{\omega_0^2 - \omega^2} \right).$$

Maximum oscillation amplitude $A_{\max} = \frac{A_0 \omega_0^2}{2\gamma \sqrt{\omega_0^2 - \gamma^2}}$ when

$$\omega = \sqrt{\omega_0^2 - 2\gamma^2}.$$



Question 1

- A mass of 0.2kg undergoes oscillations of period 1.2s on a vertically mounted spring. Determine the spring constant k .
- If the mass-spring system in (i) is initially stretched by 0.1m from equilibrium and then released, calculate the velocity of the mass after 2.0s. Sketch $x(t)$, $\dot{x}(t)$, $\ddot{x}(t)$ on separate graphs with the same time scale.

- (iii) Determine expressions for the time variation of (a) kinetic energy and (b) potential energy for the mass-spring system in (i),(ii). On the same graph, sketch kinetic energy, potential energy and total energy vs time for at least two periods.
- (iv) A rubber duck is placed in a bath. The duck has an in-built accelerometer with a USB stick memory for data logging. Analysis of the bobbing motion of the duck results in a straight line correlation between acceleration \ddot{x} (in cm/s^2) and displacement x (in cm) of the form $\ddot{x} = -223.8x$. Determine the period of the oscillation in s.
- (v) A charged ping-pong ball can be made to oscillate between two metal plates, both carrying the same charge with the same sign as the charge on the ping-pong ball. The ball has a displacement x of 2.0cm, a velocity of -3.0cm/s and an acceleration of -4.0cm/s^2 at $t = 0.0\text{s}$. Determine A, ω, ϕ and hence sketch $x(t)$ for two periods.
- (vi) A long pendulum of length l undergoes small oscillations with period $T = km^A g^B l^C$. m is the mass of the pendulum bob and g is the strength of gravity. k is a dimensionless constant.
- (a) Use dimensional analysis to find A, B, C and hence show that $T = k\sqrt{l/g}$.
- (b) As you will prove later (Q2), $k = 2\pi$. The First Church of Mars conducts commences Wednesday services with a swing of a 3.14m pendulum, starting from rest at a maximum angle of 10° . Calculate the period of the pendulum, and its maximum speed. Hence sketch $\theta(t)$ and $\dot{\theta}(t)$. $g_{\text{mars}} = 3.72\text{m/s}^2$.
- (vii) The suspension in an old car is *underdamped*. When the car hits a bump, it oscillates with frequency 5Hz. A particular bump causes a oscillation of initial amplitude 10cm, starting from rest. Assuming the damping constant $\gamma = \frac{1}{10}\omega$, determine an equation for the subsequent motion, and sketch it. What is the amplitude and velocity after 0.15s? Note: $x(t) = Ae^{-\gamma t} \cos\left(t\sqrt{\omega^2 - \gamma^2} - \phi\right)$ and $\omega = 2\pi f \times \left(1 - \frac{\gamma^2}{\omega^2}\right)^{-\frac{1}{2}}$.
- (viii) A swing at a children's playground for danger-seeking nieces of Physicists is $l = 4.2\text{m}$ long. It can be modeled as a simple pendulum of period $T = 2\pi\sqrt{l/g}$ where $g = 9.81\text{m/s}^2$. The swing is driven by a sinusoidal driving force of frequency $f = \frac{1}{2\pi}\omega$, such that the (angular) motion of the swing is described by the differential equation $\ddot{\theta} + \frac{\pi}{6T}\dot{\theta} + \left(\frac{2\pi}{T}\right)^2\theta = \left(\frac{2\pi}{T}\right)^2\frac{\pi}{36}\sin\omega t$. Use this information to sketch the amplitude and phase responses (i.e. vs ω) of the steady-state oscillation. Use the formulae for damped SHM on page 1 without proof.
- (ix) If the children's swing in (vii) was critically damped, determine the equation of motion $\theta(t)$, assuming that the swing is released from rest at $\theta = \frac{\pi}{6}$ radians. Assume the sinusoidal driving force machine has been turned off.
- (x) Show that the transient responses $x(t)$ for (a) underdamped; (b) overdamped; (c) critically damped SHM systems described on page 1 are indeed solutions to $\ddot{x} + 2\gamma\dot{x} + \omega^2x = 0$. *Tedious calculus but good for you!*

Question2 A pendulum of length l consists of a light inextensible string, supporting a small mass m . The top of the string is fixed in position, and the only significant forces on the mass are gravity (with strength g) and the tension in the string.

- (i) Draw a diagram to represent the situation and then write down Newton II in radial and tangential directions. Show that for small angles, where $\sin\theta \approx \theta$, the motion is SHM with period $\tau = 2\pi\sqrt{l/g}$.
- (ii) What will a graph of $\dot{\theta}$ vs $\omega\theta$ look like for the (SHM) motion? What would happen if there was damping? Assume the pendulum is released from rest i.e. $\theta(t) = \theta_0 \cos\omega t$.

- (iii) *Without* using the small angle approximation, use energy conservation to derive an expression of how the speed of the mass varies with θ , and sketch this. Determine an equation for the θ variation of string tension T , and sketch this too. Assume there is no damping and therefore no loss of energy.

Question 3 A charge of mass m is placed in the centre of a vacuum tube of length l . At either ends of the tube are identically charged plates, of the same sign as the charge on the mass. The force on the charge due to the left plate is k/x^2 where x is the distance from the left plate to the charge i.e. an inverse-square law.

- (i) Explain why: $m\ddot{x} = kx^{-2} - k(l-x)^{-2}$
- (ii) If $x = \frac{1}{2}l + z$, where z is a small ($|z| \ll l$) perturbation from equilibrium, show (using appropriate binomial expansions) that $\ddot{z} \approx -\frac{32k}{ml^3}z$.

Hence determine an expression for the frequency of oscillations of the charged mass m .

- (iii) Evaluate the oscillation frequency (and equivalent EM wavelength in nm) using the following atomic-scale values.

$$l = 1.23 \times 10^{-10} \text{ m}, \quad m = 1.67 \times 10^{-27} \text{ kg}, \quad k = \frac{e^2}{4\pi\epsilon_0} \quad \text{where } e = 1.602 \times 10^{-19} \text{ C}, \quad \epsilon_0 = 8.85 \times 10^{-12} \text{ Nm}^2\text{C}^{-2}.$$

Question 4 A cylinder of radius r and height h is placed in a liquid of density ρ , whereupon it partially submerges to depth $x = \alpha h$ where $0 < \alpha < 1$. What is the mass m of the cylinder in terms of ρ, r, h ?

If the cylinder is displaced by extra depth $z(t=0) = \beta h$, where $\beta \ll \alpha$, determine an expression for the period T of the ensuing oscillations, i.e. the time variation of 'extra depth' z . Assume a uniform strength of gravity g .

Determine the period T of these oscillations (in s) for a cylindrical iceberg of radius 100m and height 200m. The density of seawater is 1025 kg/m^3 and the density of ice is 920 kg/m^3 . $g = 9.81 \text{ m/s}^2$.

Question 5 *de-Moivre's theorem* states that $e^{i\theta} = \cos \theta + i \sin \theta$, which means $A \cos(\omega t - \phi)$ is the *real part* of $z = A e^{i(\omega t - \phi)}$. Consider a 'complex' version of the driven SHM equation: $\ddot{z} + 2\gamma\dot{z} + \omega_0^2 z = A_0 \omega_0^2 e^{i\omega t}$. By assuming a steady state solution $z(t) = A e^{i(\omega t - \phi)}$, substitute into the SHM equation and hence prove:

$$A = A_0 \omega_0^2 / \sqrt{(\omega_0^2 - \omega^2)^2 + 4\gamma^2 \omega^2} \quad \text{and} \quad \phi = \tan^{-1} \left(\frac{2\gamma\omega}{\omega_0^2 - \omega^2} \right).$$

Also prove that the maximum oscillation amplitude $A_{\max} = \frac{A_0 \omega_0^2}{2\gamma \sqrt{\omega_0^2 - \gamma^2}}$ when $\omega = \sqrt{\omega_0^2 - 2\gamma^2}$.

Question 6 (Adapted from a question in OCR *Mechanics 4* textbook in the *Small Oscillations & Stability* chapter).

A rigid rod of mass m and length $2a$ is suspended from a frictionless hinge. When the rod is hung in equilibrium, the bottom of the rod is a from the ground. At a point on the ground directly below the hinge is a small fixed ring. A light elastic cord of unstretched length $2a$ and spring constant $k = 5mg/2a$ is threaded through the rod and securely attached to the ring. The rod is displaced by a small angle, and oscillations of period T ensue.

- (i) Show that the moment of inertia of the rod rotating about the hinge is $I = \frac{4}{3}ma^2$.
- (ii) Show that the extension of the elastic is $x = a\sqrt{13 - 12\cos\theta}$, where θ is the angle of the rod from the vertical.
- (iii) Show that the total energy of the system is: $E = \frac{2}{3}ma^2\dot{\theta}^2 + mga(17\frac{1}{4} - 16\cos\theta)$.
- (iv) If no energy is lost (i.e. $\dot{E} = 0$) show that $T \approx 2\pi\sqrt{\frac{a}{12g}}$ for small oscillations.
- (v) Show that the period of the rod swinging *without* the elastic is exactly four times that with the elastic.